

New PID designs for sampling control and batch process optimization

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Introduction



Some facts of PID controllers

Advantages:

- Simple structure and low cost;
- Easily understood and commanded by users;
- Most widely used and commercialized in industrial applications.

Disadvantages [1, 2]:

- ◆ Generally not optimal in control performance;
(often used as an inferior to demonstrate other advanced control designs)
- ◆ Difficult to analyze the robust stability against system uncertainties.
(lowly valued for theoretical contribution in top control-relevant journals)

[1] Karl J. Åström, P.R. Kumar. Control: A perspective. *Automatica*, 2014, 50(1), 3-43.

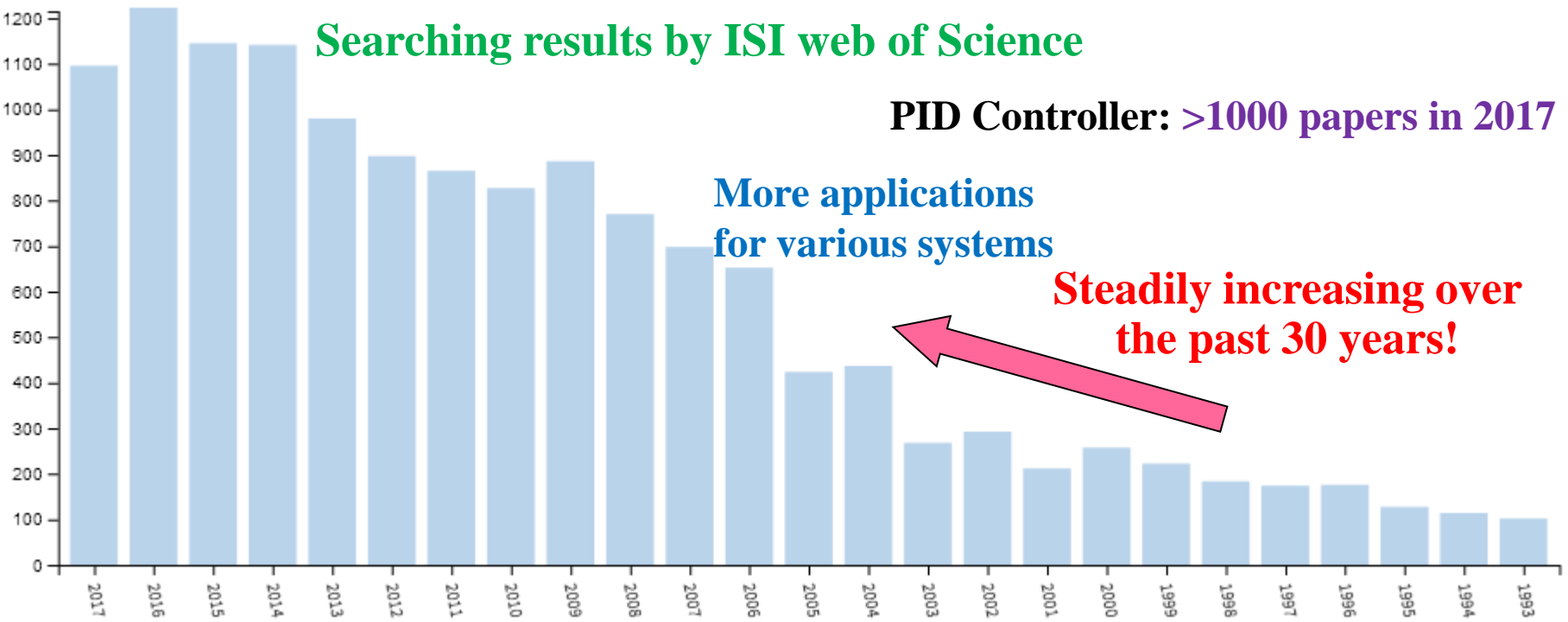
[2] Tao Liu, Furong Gao. Industrial Process Identification and Control Design: Step-test and Relay-Experiment-Based Methods. London UK: *Springer*, 2012.

Searching results by ISI web of Science

PID Controller: >1000 papers in 2017

**More applications
for various systems**

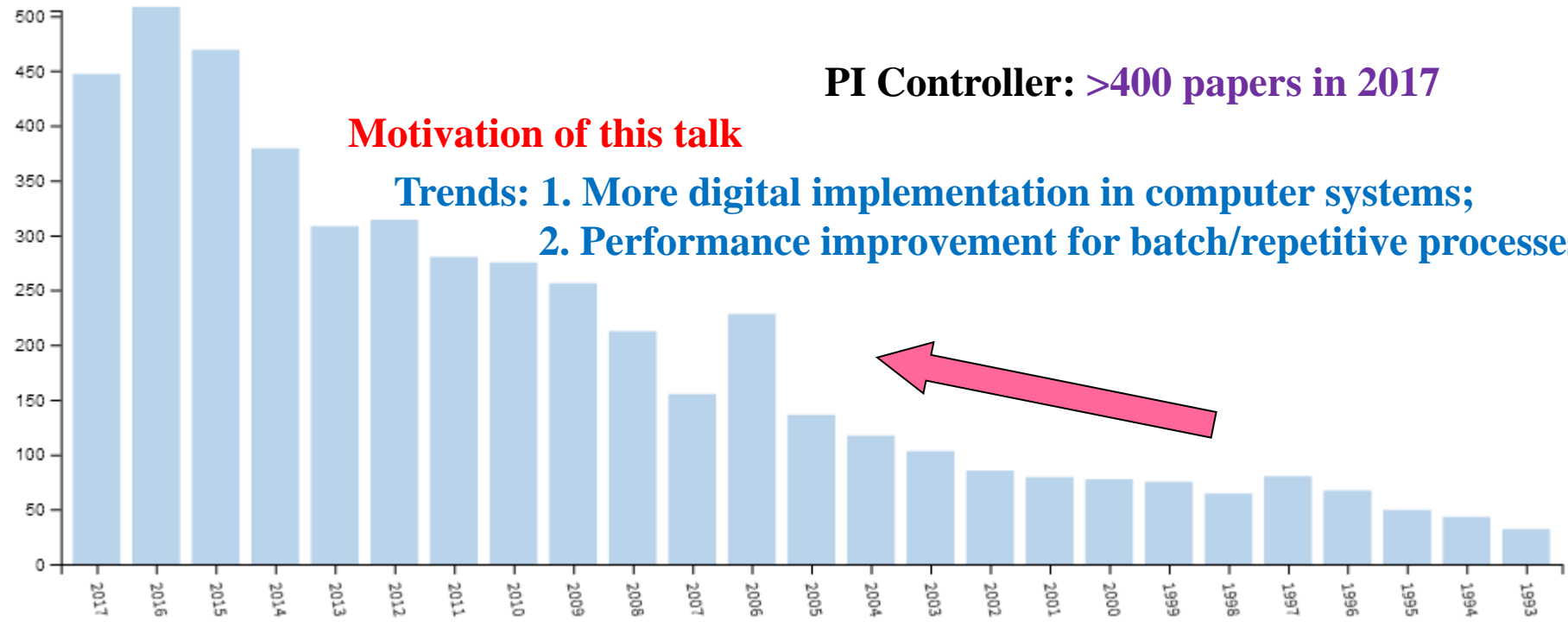
**Steadily increasing over
the past 30 years!**



PI Controller: >400 papers in 2017

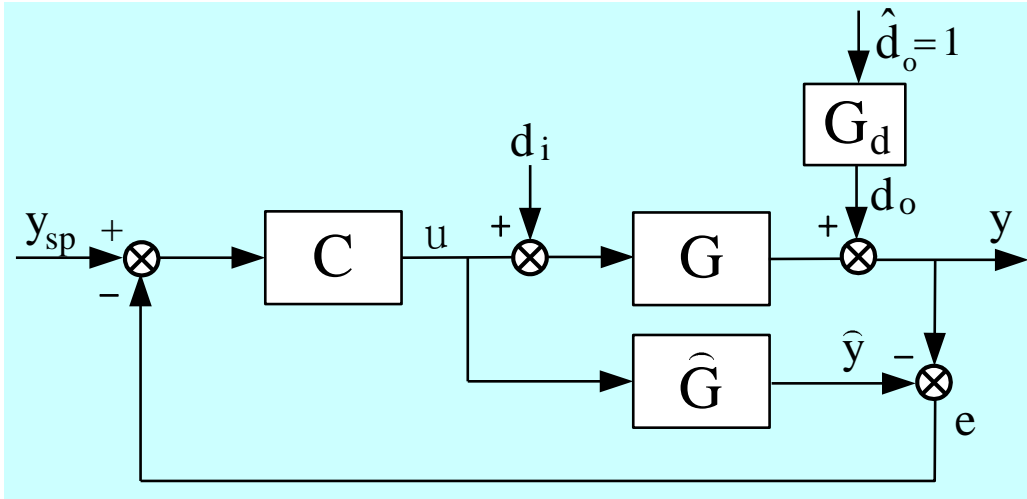
Motivation of this talk

**Trends: 1. More digital implementation in computer systems;
2. Performance improvement for batch/repetitive processes.**



PID design for sampled control systems

Relationship between the PID control and the internal model control (IMC)



Process model:

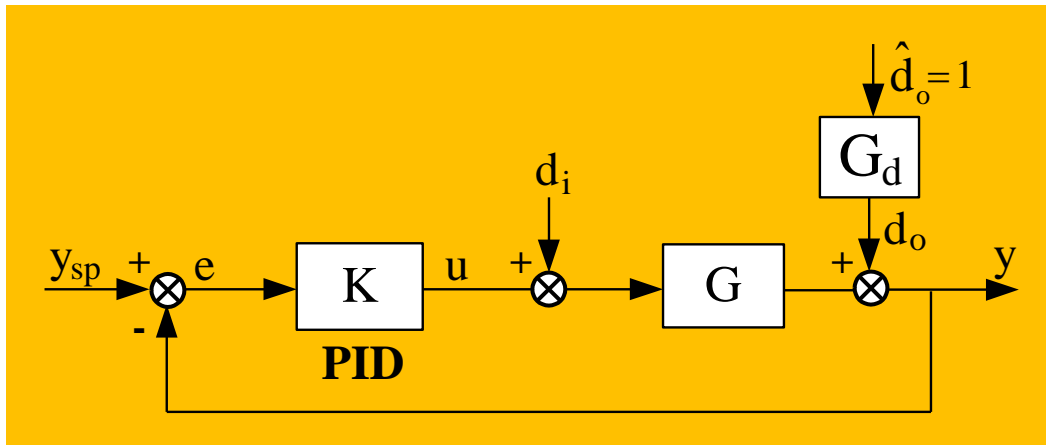
$$\hat{G} = \frac{B(z)}{A(z)} z^{-d}$$

Frequency domain:

$$\hat{G} = \frac{B(s)}{A(s)} e^{-\theta s}$$

IMC: applicable for stable processes

The internal model control (IMC) structure



The unity feedback control structure

Equivalent relationship

$$K = \frac{C}{1 - \hat{G}C}$$

PID: applicable for stable, integrating, and unstable processes

Review of the IMC design in frequency domain

Step 1. Decompose the model into the minimum-phase (MP), non-MP, and all-pass parts.

For example:
$$\widehat{G} = \frac{k_p(-\tau_0 s + 1)}{(\tau_1 s + 1)(\tau_2 s + 1)} e^{-\theta s}$$

where $\tau_0 > 0$, $\tau_1 > 0$, $\tau_2 > 0$.

$$\widehat{G} = \widehat{G}_{\text{mp}} \widehat{G}_{\text{nmp}} \widehat{G}_{\text{ap}} \quad \widehat{G}_{\text{ap}} = e^{-\theta s}$$

$$\widehat{G}_{\text{mp}} = \frac{k_p}{(\tau_1 s + 1)(\tau_2 s + 1)} \quad \widehat{G}_{\text{nmp}} = -\tau_0 s + 1$$

Step 2. Specify the desired closed-loop transfer function for set-point tracking.

$$T = F_1 F_2 \widehat{G}_{\text{nmp}} \widehat{G}_{\text{ap}}$$

where F_1 and F_2 are two low-pass filters.

For a stable process described as above,

$$F_1 = \frac{1}{(\lambda s + 1)^3} \quad \rightarrow \quad \text{ensure } F_1 \widehat{G}_{\text{mp}} \text{ strictly proper}$$

$$F_2 = \frac{1}{\tau_0 s + 1} \quad \rightarrow \quad \text{ensure } F_2 \widehat{G}_{\text{nmp}} \text{ all-pass, i.e. } \frac{-\tau_0 s + 1}{\tau_0 s + 1} = e^{-2\tau_0 s}$$

Desired closed-loop transfer function

$$T = \frac{1}{(\lambda s + 1)^3} \frac{-\tau_0 s + 1}{\tau_0 s + 1} e^{-\theta s}$$

single tuning parameter λ

No overshoot
Minimal ISE

Discrete-time domain IMC design

Step 1. Decompose the model into the minimum-phase (MP), non-MP, and all-pass parts.

For example:
$$\widehat{G} = \frac{k_p(z-b_1)(z-b_2)}{(z-a_1)(z-a_2)} z^{-d}$$

where $|a_1| < 1, |a_2| < 1, |b_1| < 1, |b_2| > 1$.

$$\widehat{G} = \widehat{G}_{\text{mp}} \widehat{G}_{\text{nmp}} \widehat{G}_{\text{ap}} \quad \widehat{G}_{\text{ap}} = z^{-d}$$

$$\widehat{G}_{\text{mp}} = \frac{k_p(z-b_1)}{(z-a_1)(z-a_2)} \quad \widehat{G}_{\text{nmp}} = z - b_2$$

Step 2. Specify the desired closed-loop transfer function for set-point tracking.

$$T = F_1 F_2 \widehat{G}_{\text{nmp}} \widehat{G}_{\text{ap}}$$

where F_1 and F_2 are two low-pass filters.

For a stable process described as above,

$$F_1 = \frac{(1-\lambda_c)^2}{(z-\lambda_c)^2} \quad \Rightarrow \quad \text{ensure } F_1 \widehat{G}_{\text{mp}} \text{ strictly proper}$$

$$F_2 = \frac{1-b_2^{-1}}{(1-b_2)(z-b_2^{-1})} \quad \Rightarrow \quad \text{ensure } F_2 \widehat{G}_{\text{nmp}} \text{ all-pass, i.e. } \frac{(1-b_2^{-1})(z-b_2)}{(1-b_2)(z-b_2^{-1})}$$

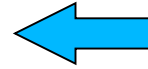
Desired closed-loop transfer function

$$T = \frac{(1-\lambda_c)^2}{(z-\lambda_c)^2} \frac{(1-b_2^{-1})(z-b_2)}{(1-b_2)(z-b_2^{-1})} z^{-d}$$

Discrete-time domain IMC design

Step 3. Determine the IMC controller.

$$C_{\text{IMC}}(z) = \frac{T(z)}{\widehat{G}(z)}$$



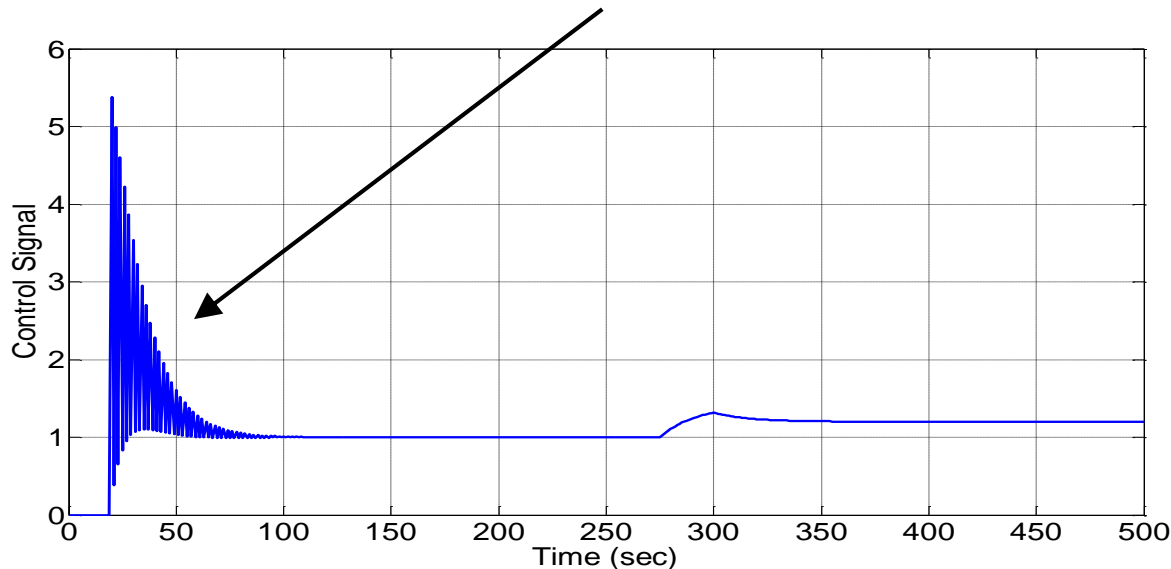
$$T = GC$$

For a stable process described as above,

single tuning parameter λ_c

$$C_{\text{IMC}}(z) = \frac{(z - a_1)(z - a_2)(1 - \lambda_c)^2}{k_p(z - b_1)(z - \lambda_c)^2} \frac{(1 - b_2^{-1})}{(1 - b_2)(z - b_2^{-1})}$$

In case $-1 < b_1 < 0$, it could provoke **inter-sample ringing** in the control signal !



Discrete-time domain IMC design

Solution: Introduce another filter to remove such a zero for implementation

$$F_3(z) = z^{-1} \frac{z - b_1}{1 - b_1} \quad \rightarrow \quad C_{\text{RIMC}}(z) = F_3(z) C_{\text{IMC}}(z)$$

Control performance assessment

For a stable process described by

$$G_1(z) = \frac{K_{p1}}{z - z_p} z^{-d}$$

$$T_d(z) = \frac{1 - \lambda_c}{z - \lambda_c}$$



$$Y(z) = \frac{1 - \lambda_c}{z - \lambda_c} z^{-d} R(z)$$



$$R(z) = \frac{z}{z - 1}$$

Step response in time domain

$$y(kT_s) = \begin{cases} 0, & k \leq d, \\ 1 - \lambda_c^{k-d}, & k > d. \end{cases}$$

Set-point tracking error

$$IAE_r = \sum_{k=d+1}^{\infty} [1 - y(kT_s)] = \sum_{k=d+1}^{\infty} \lambda_c^{k-d} = \lim_{n \rightarrow \infty} \frac{\lambda_c (1 - \lambda_c^n)}{1 - \lambda_c} = \frac{\lambda_c}{1 - \lambda_c}$$

Disturbance rejection error to a step change

$$IAE_d = \sum_{k=d+1}^{\infty} y(kT_s) = \frac{K_{p1}}{(1 - \lambda_c)(1 - z_p)}$$

Quantitative tuning

IMC-based PID design in frequency domain

Step 4. Determine the equivalent controller in a unity feedback control structure.

$$K = \frac{C}{1 - \widehat{G}C}$$

For $\widehat{G} = \frac{k_p(-\tau_0 s + 1)}{(\tau_1 s + 1)(\tau_2 s + 1)} e^{-\theta s}$ \rightarrow $K = \frac{(\tau_1 s + 1)(\tau_2 s + 1)}{k_p [(\lambda s + 1)^3 (\tau_0 s + 1) - (-\tau_0 s + 1)e^{-\theta s}]}$

Step 5. Determine the PID controller by the Taylor approximation [1].

Since $\lim_{s \rightarrow 0} K(s) = \infty$, let $K(s) = \frac{M}{s}$

$$K(s) = \frac{1}{s} \left[M(0) + M'(0)s + \frac{M''(0)}{2!} s^2 + \dots \right]$$

**Important advantage:
single tuning parameter**

$$K(s) = k_C + \frac{1}{\tau_I s} + \frac{\tau_D s}{\tau_F s + 1}$$

$$\begin{cases} k_C = M'(0) \\ \tau_I = 1/M(0) \\ \tau_D = M''(0)/2 \end{cases}$$

$$\tau_F = (0.01 - 0.1)\tau_D$$

IMC-based PID design in frequency domain

Alternatively, the delay term in the process model or the IMC controller may be rationally approximated to derive a PID, e.g.,

The first-order Taylor approximation [2]: $e^{-\theta s} \approx 1 - \theta s$

Pade approximation [3]: $e^{-\theta s} \approx (1 - \frac{\theta}{2}s) / (1 + \frac{\theta}{2}s)$

Model reduction [4, 5]: $\frac{1}{(\tau_1 s + 1)(\tau_2 s + 1)} \approx \frac{1}{(\tau_1 + \frac{\tau_2}{2})s + 1} e^{-\frac{\tau_2}{2}s}$ ‘Half rule’

Frequency response fitting [6]: $\min \sum_{i=1}^m |K_{\text{PID}}(j\omega_i) - K(j\omega_i)|$ $\omega_i \in (0.1, 1)\omega_{\text{cb}}$

[2] Rivera, D. E., Morari, M., Skogestad, S. Internal model control. 4. PID controller design. *Ind. Eng. Chem. Res.*, 25, 252-265, 1986.

[3] Fruehauf P. S., Chien I.-L., Lauritsen M. D. Simplified IMC-PID tuning rules. *ISA Transactions*, 1994, 33, 43-59.

[4] Skogestad, S. Simple analytical rules for model reduction and PID controller tuning. *Journal of Process Control*, 2003, 13, 291-309.

[5] Lee, J. Cho, W. Edgar, T. F. Simple analytic PID controller tuning rules revisited, *Industrial & Engineering Chemistry Research*, 53(13), 5038-5047, 2014.

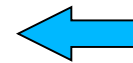
[6] Wang, Q. G., Yang, X. P. Single-loop controller design via IMC principles. *Automatica*, 37, 2041-2048, 2001.

IMC-based PID design in discrete-time domain

For sampling control implementation, the first-order differentiation is generally used to discretize the above frequency domain design.

For direct PID design in discrete-time domain, let the equivalent controller in a unity feedback control structure be

$$K(z) = \frac{M(z)}{z-1}$$



$$K = \frac{C}{1-\widehat{GC}}$$

owing to $\lim_{z \rightarrow 1} K(z) = \infty$.

A PID controller is determined by the Taylor approximation [1, 2].

$$K(z) = \frac{1}{z-1} \left[M(1) + M'(1)(z-1) + \frac{M''(1)}{2!} (z-1)^2 + \dots \right]$$

$$K_{\text{PID}}(z) = k_c \left[1 + \frac{1}{\tau_I(z-1)} + (1 - \alpha\tau_D) \frac{\tau_D(z-1)}{z - \alpha\tau_D} \right]$$

$$\begin{cases} k_c = M'(1) \\ \tau_I = M'(1) / M(1) \\ \tau_D = M''(1) / 2M'(1) \end{cases}$$

single tuning parameter λ

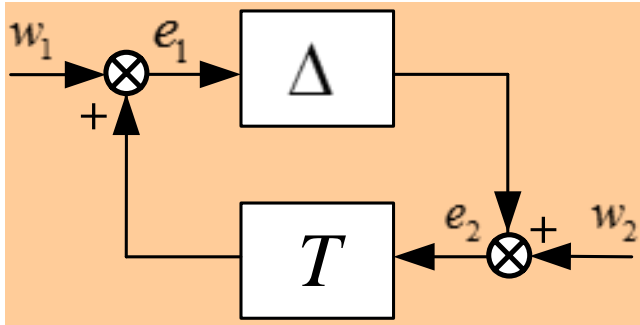
$$\alpha \in (0.01, 0.1)$$

[1] Wang, D., Liu T.*, Sun X., Zhong Chongquan. Discrete-time domain two-degree-of-freedom control for integrating and unstable processes with time delay. *ISA Transactions*, 63, 121-132, 2016.

[2] Cui J., Chen Y., Liu T.*. Discrete-time domain IMC-based PID control design for industrial processes with time delay. *The 35th Chinese Control Conference (CCC)*, Chengdu, China, 5946-5951, 2016.

IMC-based PID design in discrete-time domain

Closed-loop system robust stability analysis



The $T-\Delta$ structure

Small gain theorem



$$\|T(z)\Delta_m(z)\|_\infty < 1$$

$$T(z) = \frac{G_p(z)C_{\text{PID}}(z)}{1 + G_p(z)C_{\text{PID}}(z)}$$

$$\Delta_m(z) = [G(z) - G_m(z)]/G_m(z)$$

Magnitude/Phase margin, Maximal sensitivity index M_s

For a stable process described by

$$G_1(z) = \frac{k_p}{z - \tau} z^{-d}$$

No analytical solution!



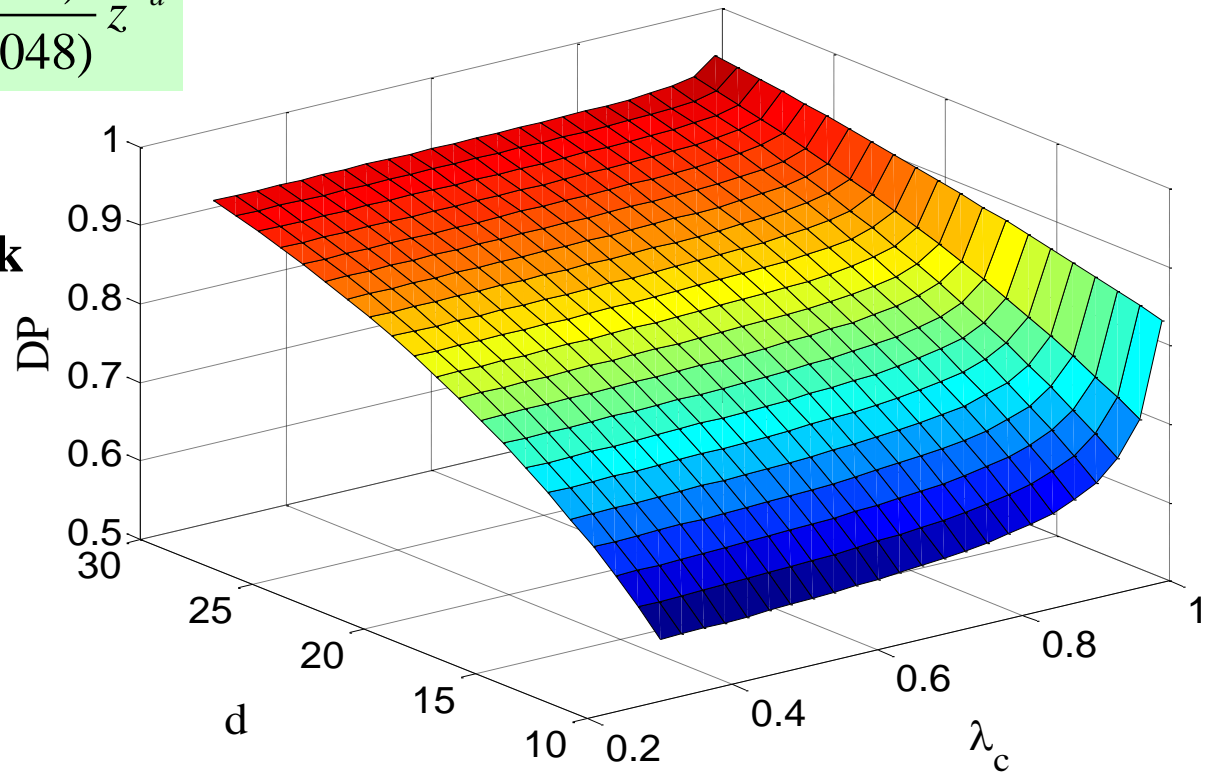
$$\left\| \frac{\frac{k_p}{z - \tau} z^{-d} K \left[1 + \frac{1}{T_i(z-1)} + (1 - \alpha T_d) \frac{T_d(z-1)}{z - \alpha T_d} \right]}{1 + \frac{k_p}{z - \tau} z^{-d} K \left[1 + \frac{1}{T_i(z-1)} + (1 - \alpha T_d) \frac{T_d(z-1)}{z - \alpha T_d} \right]} \right\|_\infty < \frac{1}{\|\Delta_m(z)\|_\infty}$$

IMC-based PID design in discrete-time domain

Disturbance rejection performance

$$G(z) = \frac{0.0065904(z + 0.9222)}{(z - 0.8669)(z - 0.9048)} z^{-d}$$

Disturbance response peak



Tuning guideline: The single adjustable parameter λ_c can be monotonically increased or decreased in a range of $\lambda_c \in (0.8, 1)$ to meet a good trade-off between the closed-loop control performance and its robust stability.

IMC-based PID design in discrete-time domain

An illustrative example:

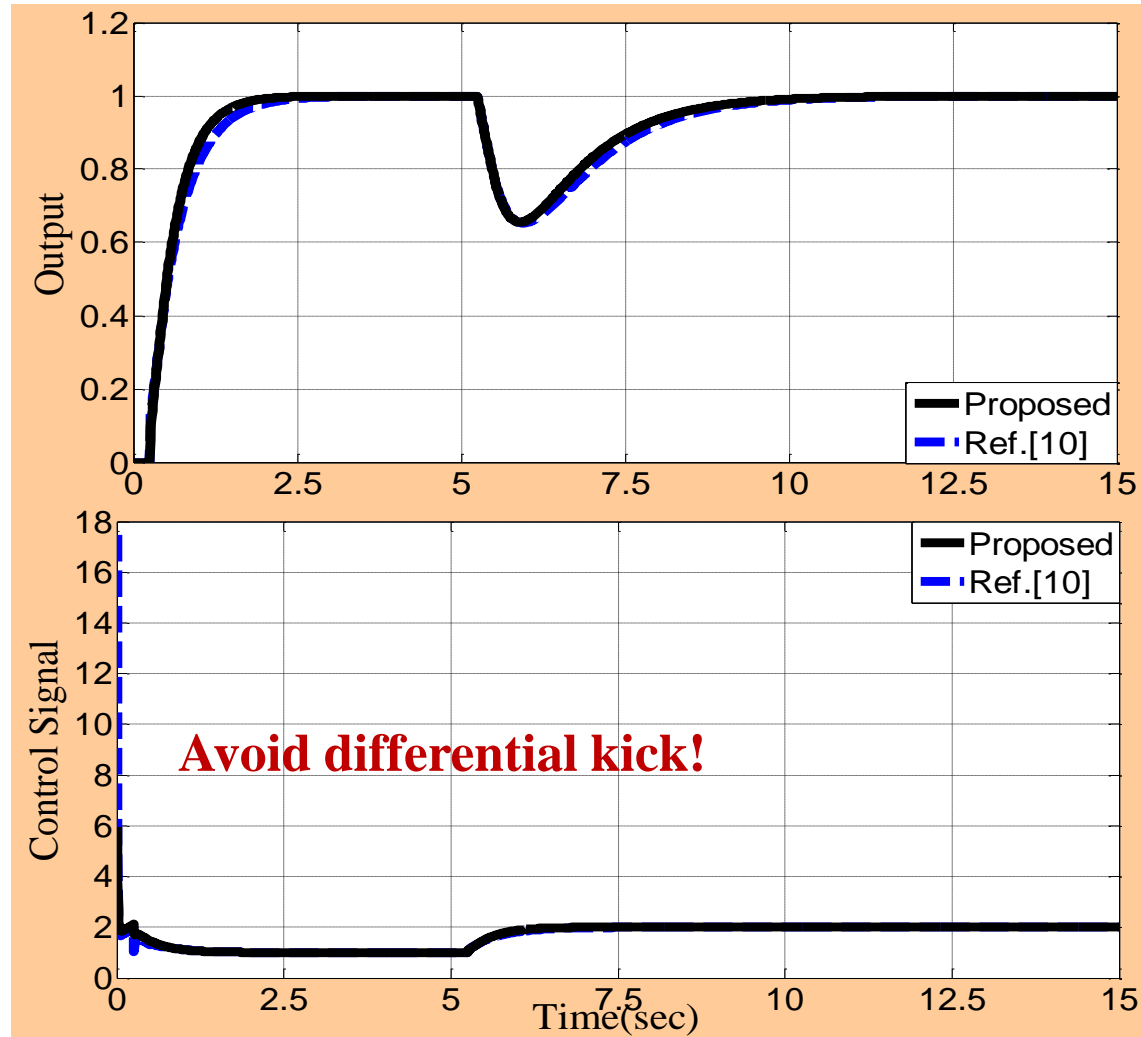
$$G(s) = \frac{1}{s+1} e^{-0.25s}$$

Sampling period: $T_s = 0.02(s)$

$$G(z) = \frac{0.0003947(z + 0.9868)}{(z-1)(z-0.9608)} z^{-25}$$

$$C_{\text{IMC}}(z) = \frac{(z-0.99)(1-\lambda_c)}{0.00995(z-\lambda_c)}$$

$$C_{\text{PID}}(z) = 1.7049 \left[1 + \frac{1}{105.2388(z-1)} + \frac{2.4956(z-1)}{z-0.4791} \right]$$



Improved IMC-based PID design for disturbance rejection

For a slow process described by $G = \frac{k_p e^{-s}}{\tau_p s + 1}$ with a large time constant τ_p ,

Discrete-time domain model: $G(z) = \frac{k_p}{z - z_p} z^{-d}$ with a pole $|z_p| < 1$ close to the unit circle

Load disturbance transfer function
(optimal $1 - GC = 1 - T$ by IMC) $\frac{y}{d_i} = G(1 - GC)$ $\frac{y}{\hat{d}_o} = G_d(1 - GC)$

The slow pole of G or G_d affects the disturbance rejection performance !

Solution: Introduce an asymptotic constraint to remove the effect of slow dynamics.

Modify the desired transfer function: $T_{\text{RIMC}} = \frac{(\alpha s + 1)e^{-\theta s}}{(\lambda_f s + 1)^2}$ $T_d(z) = \frac{(1 - \lambda_c)^{n_d} (\beta_0 + \beta_1 z)}{(z - \lambda_c)^{n_d}}$

$$\alpha = \tau_p \left[1 - \left(\frac{\lambda_f}{\tau_p} - 1 \right)^2 e^{-\frac{\theta}{\tau_p}} \right]$$

$$\lim_{s \rightarrow -1/\tau_p} (1 - T_{\text{RIMC}}) = 0$$

Discrete-time domain: $\lim_{z \rightarrow 1} (1 - T_d) = 0$ $\lim_{z \rightarrow -1/z_p} (1 - T_d) = 0$

$$\begin{cases} \beta_1 = \frac{(z_p - \lambda_c)^{n_d} - (1 - \lambda_c)^{n_d}}{(z_p - 1)(1 - \lambda_c)^{n_d}} \\ \beta_0 = 1 - \beta_1 \end{cases}$$

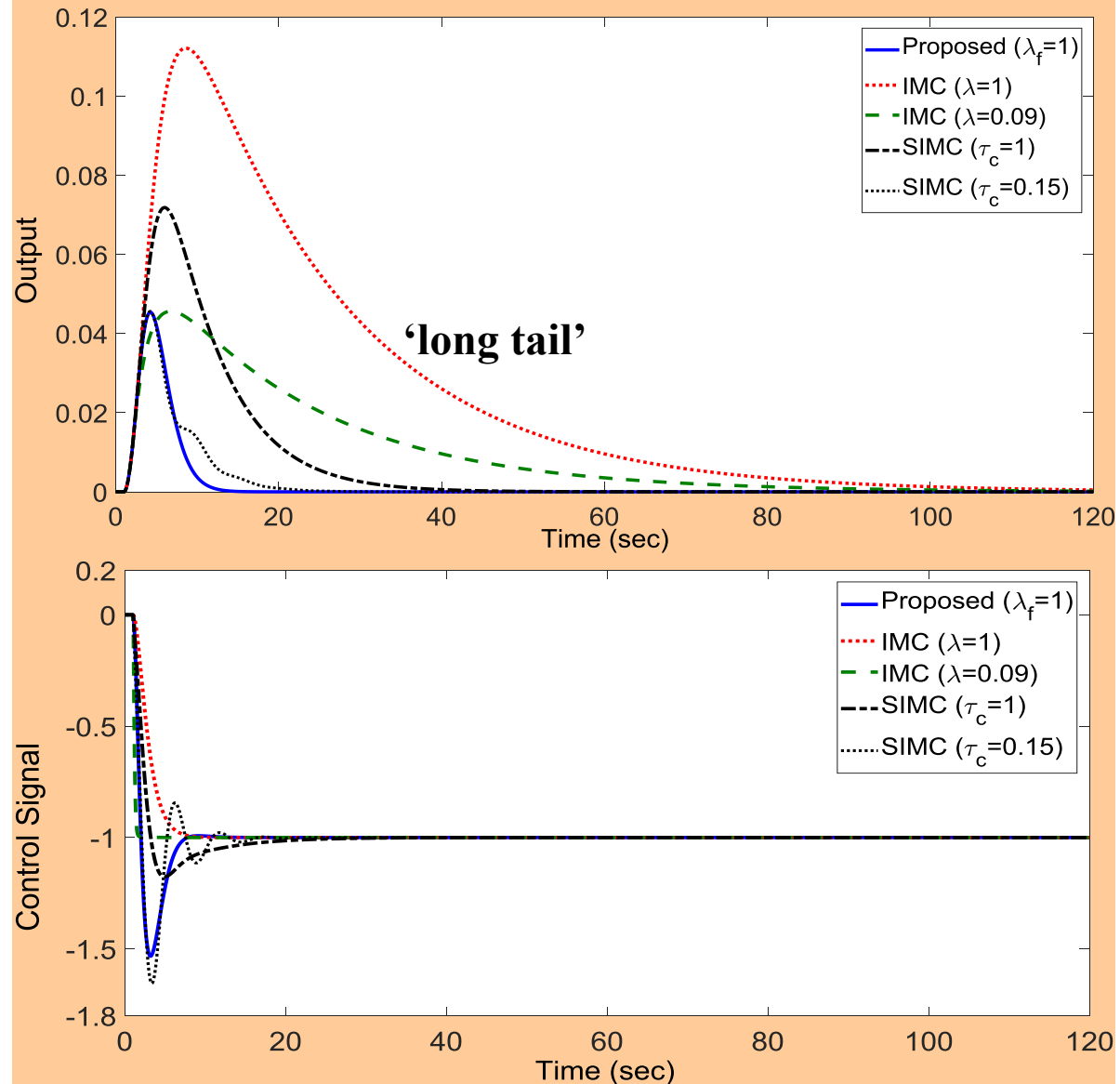
Improved IMC-based PID design for disturbance rejection

An illustrative example:

$$G = \frac{e^{-s}}{(20s+1)(2s+1)}$$

Slow pole: $s = -\frac{1}{20}$

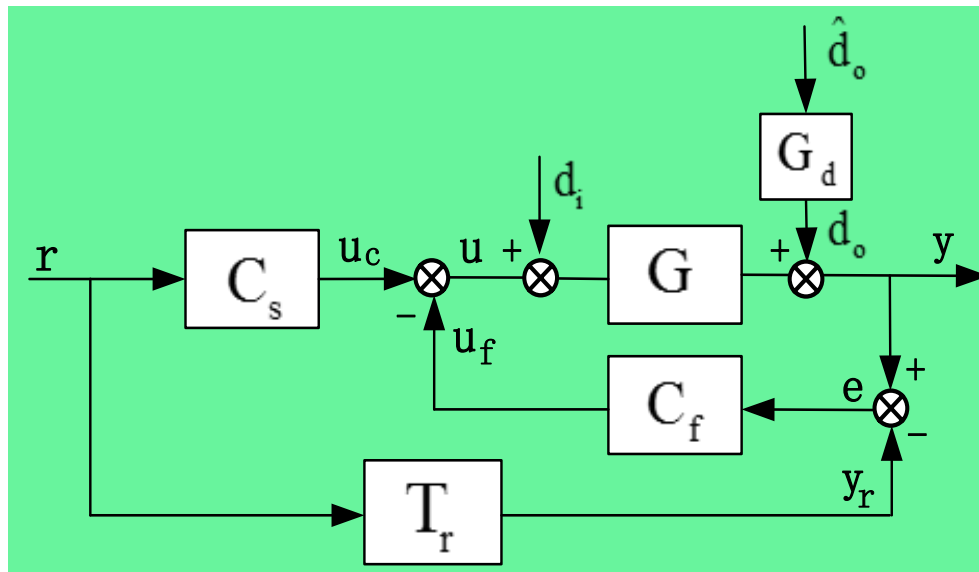
Comparison with the standard IMC and SIMC by Skogestad (JPC, 2003)



Discrete-time PID design for integrating and unstable processes

Problem: The classical PID control structure could not suppress large overshoot for set-point tracking ! There exists severe water-bed effect!

Solution: A two-degree-of-freedom (2DOF) control structure



Advantage: the set-point tracking is decoupled from load disturbance rejection.

Set-point tracking: open-loop control

IMC design:

$$T_r = GC_s$$

Disturbance rejection: closed-loop control using a PID controller, C_f

The desired transfer function for set-point tracking is the same as the standard IMC design

The desired transfer function for disturbance rejection :

$$T_d(z) = (\beta_0 + \beta_1 z + \dots + \beta_m z^m) \frac{(1 - \lambda_f)^{m+1}}{(z - \lambda_f)^{m+1}} z^{-d}$$

Discrete-time PID design for integrating and unstable processes

Integrating process:

$$\widehat{G}_I(z) = \frac{k_p(z - z_0)}{(z - 1)(z - z_p)} z^{-d}$$

Unstable process:

$$\widehat{G}_U(z) = \frac{k_p(z - z_0)}{(z - z_u)(z - z_p)} z^{-d}$$

The following asymptotic tracking constraints must be satisfied,

$$\lim_{z \rightarrow 1} (1 - T_d) = 0$$

$$\lim_{z \rightarrow \eta} (1 - T_d) = 0 \quad \eta = z_u \text{ or } \eta = z_p \text{ (close to the unit circle)}$$

$$\lim_{z \rightarrow 1} \frac{d}{dz} (1 - T_d) = 0$$

$$T_d = \frac{\widehat{G}C_f}{1 + \widehat{G}C_f}$$



Derive the closed-loop controller:

$$C_f = \frac{T_d}{1 - T_d} \cdot \frac{1}{G}$$

Taylor approximation



PID

Discrete-time PID design for integrating and unstable processes

Integrating process:

$$G(s) = \frac{0.1e^{-5s}}{s(5s+1)}$$

Frequency domain design:

$$C_s(s) = \frac{s(5s+1)}{0.1(\lambda_c s+1)^2}$$

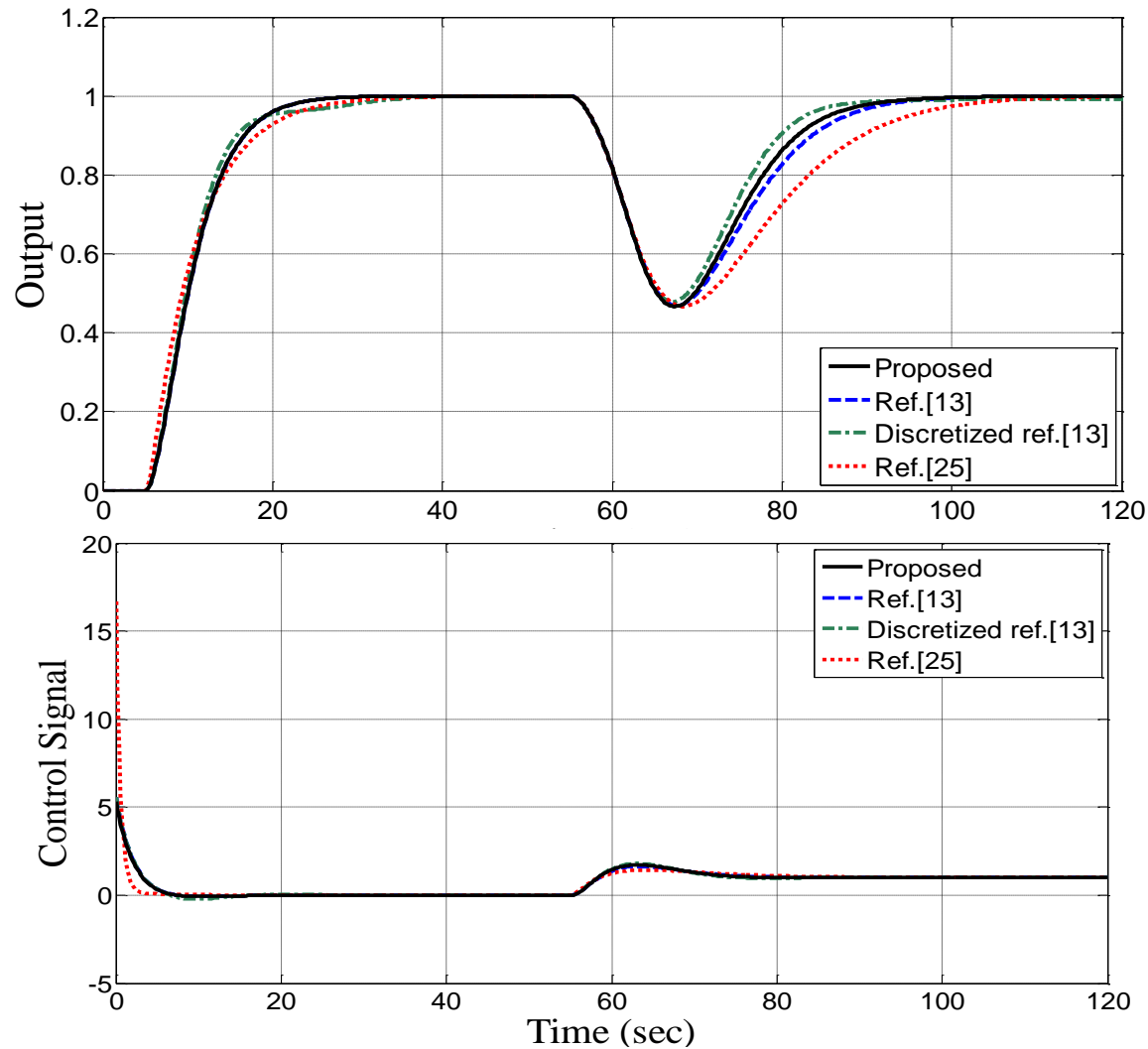
$$T_r(s) = \frac{e^{-5s}}{(\lambda_c s+1)^2}$$

$$C_f(s) = \frac{1}{G(s)} \cdot \frac{T_d(s)}{1-T_d(s)}$$

$$T_d(s) = \frac{\eta_2 s^2 + \eta_1 s + 1}{(\lambda_f s+1)^4} e^{-5s}$$

$$\eta_1 = 4\lambda_f + 5$$

$$\eta_2 = 5\eta_1 + 25(0.2\lambda_f - 1)^4 e^{-1} - 25$$



[1] Wang, D., Liu T.*, Sun X., Zhong Chongquan. Discrete-time domain two-degree-of-freedom control for integrating and unstable processes with time delay. *ISA Transactions*, 63, 121-132, 2016.

[13] Liu T.*, Gao, F. Enhanced IMC design of load disturbance rejection for integrating and unstable processes with slow dynamics. *ISA Transactions*, 50 (2), 239-248, 2011.

Discrete-time PID design for integrating and unstable processes

Sampling period: $T_s = 0.2(s)$

$$G(z) = \frac{0.0003947(z + 0.9868)}{(z - 1)(z - 0.9608)} z^{-25}$$

$$C_s(z) = \frac{(z - 1)(z - 0.9608)(1 - \lambda_c)^2}{0.0007842(z - \lambda_c)^2}$$

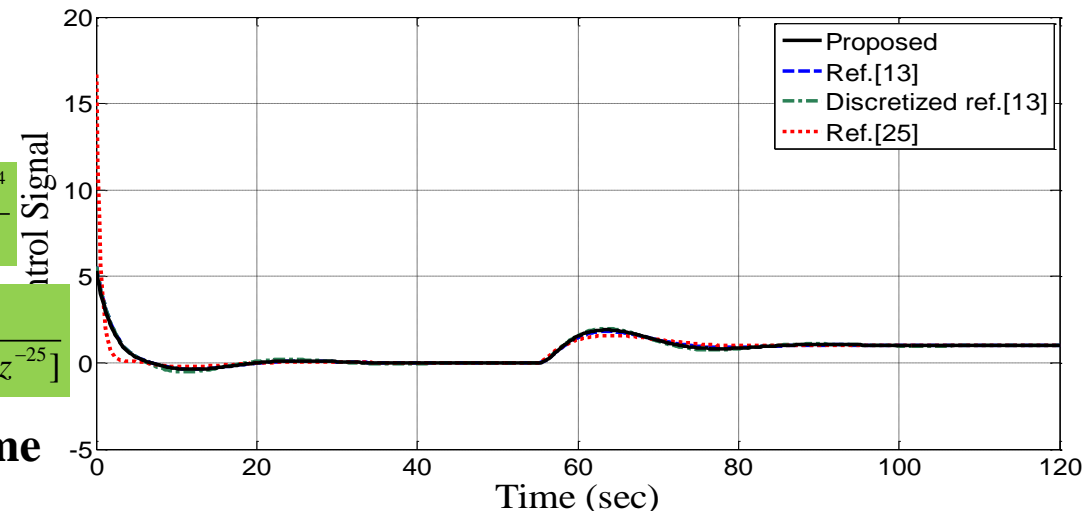
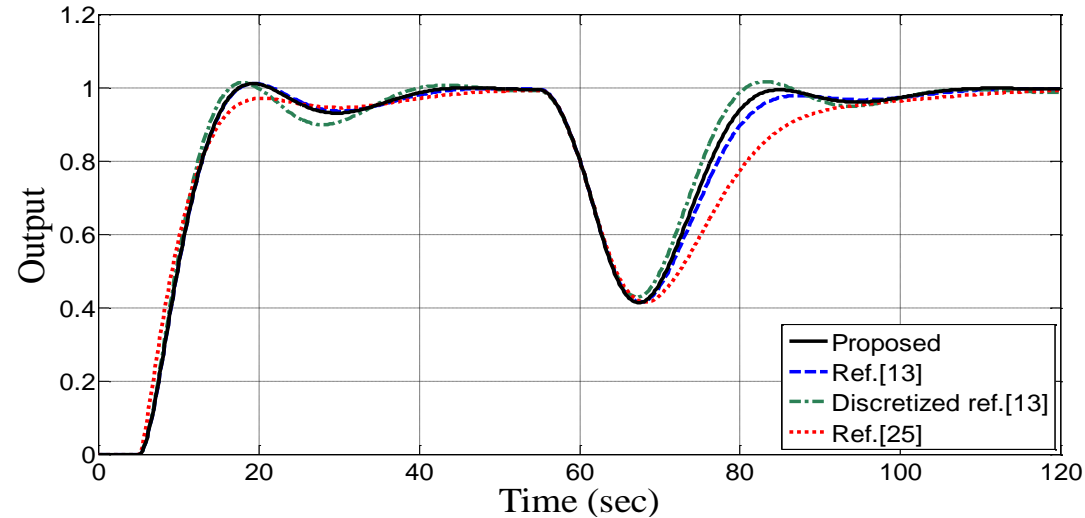
$$T_d(z) = \frac{(\beta_0 + \beta_1 z + \beta_2 z^2)(1 - \lambda_f)^4}{(z - \lambda_f)^4} z^{-25}$$

$$\beta_1 = d + \frac{4}{1 - \lambda_f} - 2\beta_2$$

$$\beta_2 = \frac{(\tau_p - \lambda_f)^4 \tau_p^d + 4(1 - \lambda_f)^3(1 - \tau_p) + (d - d\tau_p - 1)(1 - \lambda_f)^4}{(1 - \lambda_f)^4(\tau_p - 1)^2}$$

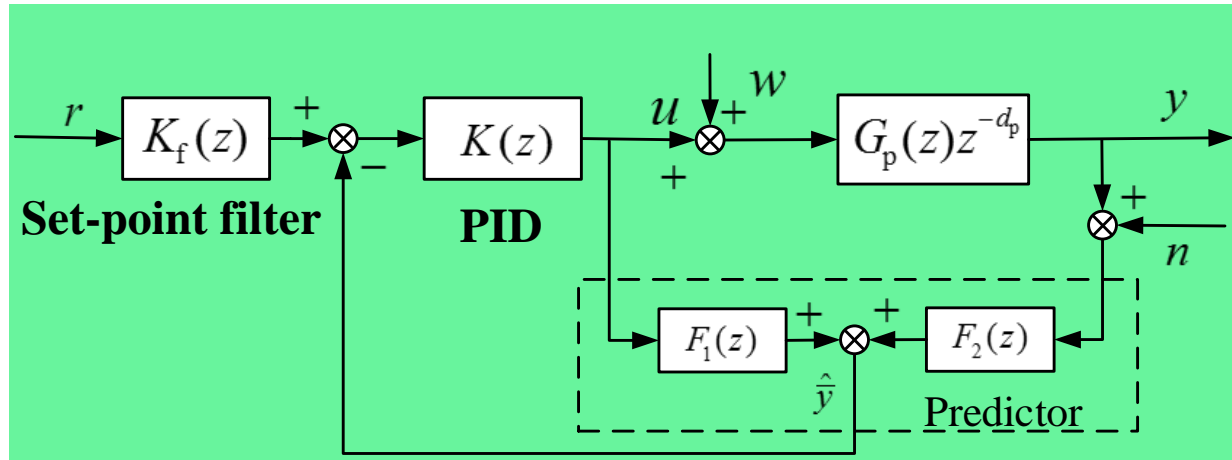
$$C_f(z) = \frac{(1 - \lambda_f)^4(\beta_0 + \beta_1 z + \beta_2 z^2)(z - 1)(z - \tau_p)}{k_p z(1 - z_0)[(z - \lambda_f)^4 - (\beta_0 + \beta_1 z + \beta_2 z^2)(1 - \lambda_f)^4 z^{-25}]}$$

Perturbation: the process gain and time constant are actually 20% larger

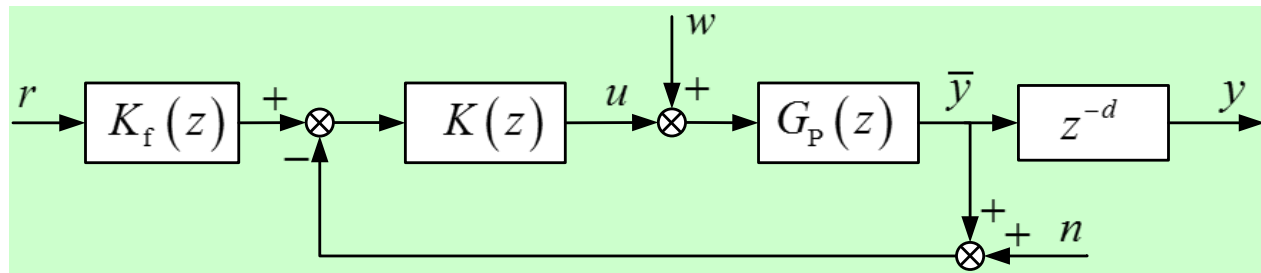


Discrete-time PID design for long time delay processes

Predictor based control structure



Delay-free output prediction



[1] **Tao Liu***, Pedro García, Yueling Chen, Xuhui Ren, Pedro Albertos, Ricardo Sanz. New Predictor and 2DOF Control Scheme for Industrial Processes with Long Time Delay. *IEEE Transactions on Industrial Electronics*, 65 (5), 4247-4256, 2018.

[2] Yueling Chen, **Tao Liu***, Pedro García, Pedro Albertos. Analytical design of a generalized predictor-based control scheme for low-order integrating and unstable systems with long time delay. *IET Control Theory & Application*, 10(8), 884-893, 2016.

Discrete-time PID design for long time delay processes

The nominal system transfer function

$$y(z) = K_f(z) \frac{K(z)G_p(z)}{1 + K(z)\hat{G}(z)} z^{-d_p} r(z) + \frac{G_p(z)}{1 + K(z)\hat{G}(z)} [1 + K(z)F_1(z)] z^{-d_p} w(z)$$

Closed-loop transfer function for disturbance rejection

$$\frac{u(z)}{w(z)} = T_d(z) = \frac{K(z)G(z)}{1 + K(z)G(z)}$$

The desired closed-loop transfer function (**free of time delay**)

$$T_d(z) = \frac{(1 - \lambda_c)^{n_d}}{(z - \lambda_c)^{n_d}} \sum_{i=0}^l \beta_i z^i$$

$$\sum_{i=0}^l \beta_i = 1$$



$$K(z) = \frac{T_d(z)}{1 - T_d(z)} \cdot \frac{1}{G_p(z)}$$

Taylor approximation



PID

Discrete-time PID design for long time delay processes

Example: $G(s) = \frac{e^{-27.5s}}{52.5s + 1}$

Sampling period: $T_s = 0.5(s)$

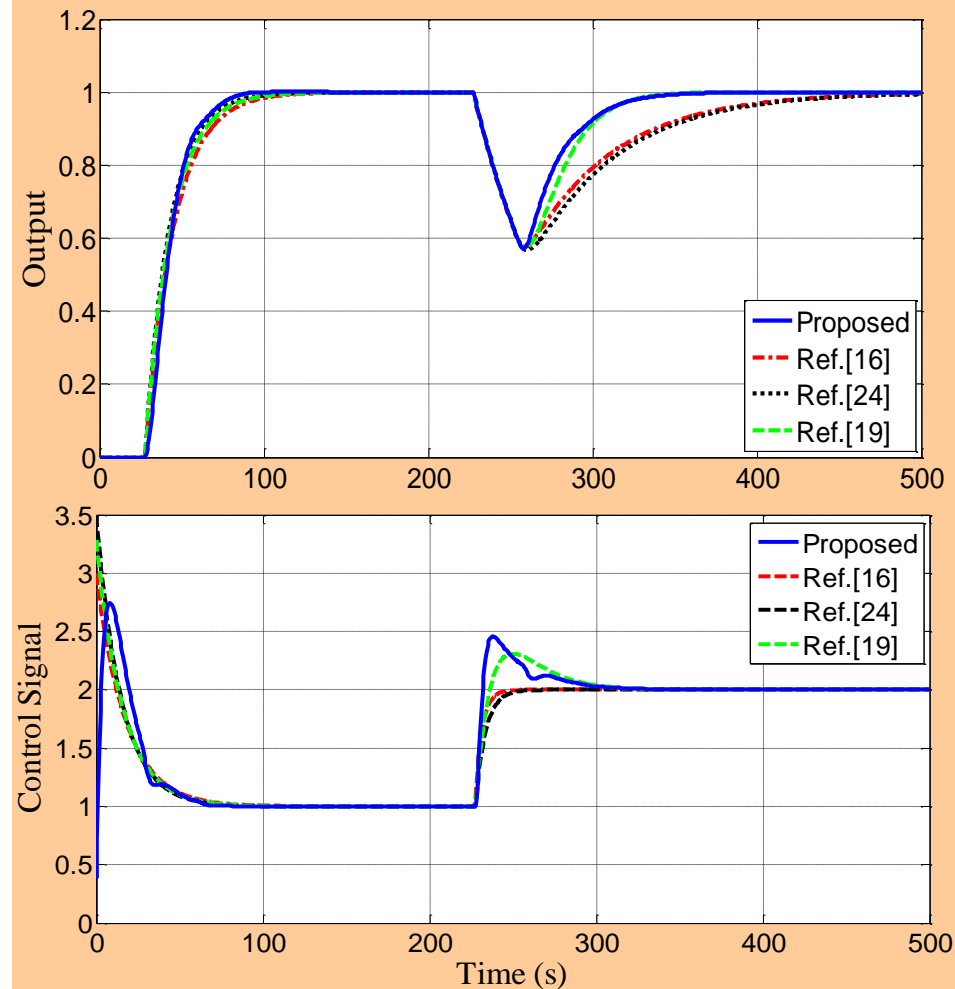
$$G(z) = \frac{0.009479}{z - 0.9905} z^{-55}$$

PI controller: $K(z) = \frac{(1 - \lambda_c)^2 (\beta_1 z + \beta_0)}{0.009479(z - 1)}$

$$\beta_0 = 1 - \beta_1$$

$$\beta_1 = (z_p - \lambda_c)^2 - (1 - \lambda_c)^2 / (z_p - 1)(1 - \lambda_c)^2$$

$$K_f(z) = \frac{(z - \lambda_c)^2 (1 - \lambda_f)^2 z}{(1 - \lambda_c)^2 (\beta_0 + \beta_1 z)(z - \lambda_f)^2}$$



[16] Kirtania, K.M. Choudhury, A.A.S. A novel dead time compensator for stable processes with long dead times. *J. Process Control*, 22(3), 612-625, 2012.

[24] Zhang, W., Rieber, J.M., Gu, D. Optimal dead-time compensator design for stable and integrating processes with time delay. *J. Process Control*, 18(5), 449-457, 2008.

[19] Normey-Rico J.E., Camacho E. F. Unified approach for robust dead-time compensator design. *J. Process Control*, 19: 38-47, 2009.

Discrete-time PID design for long time delay processes

A temperature control system of a 4-liter jacketed reactor for pharmaceutical crystallization

Sampling period: $T_s = 3(s)$

Step response identification:

Integrating type process model: $G(s) = \frac{0.0004529}{s(760.40s + 1)} e^{-100.25s}$

$$G(z) = \frac{2.6765 \times 10^{-6} (z + 0.9989)}{(z - 1)(z - 0.9961)} z^{-34}$$

Closed-loop controller:

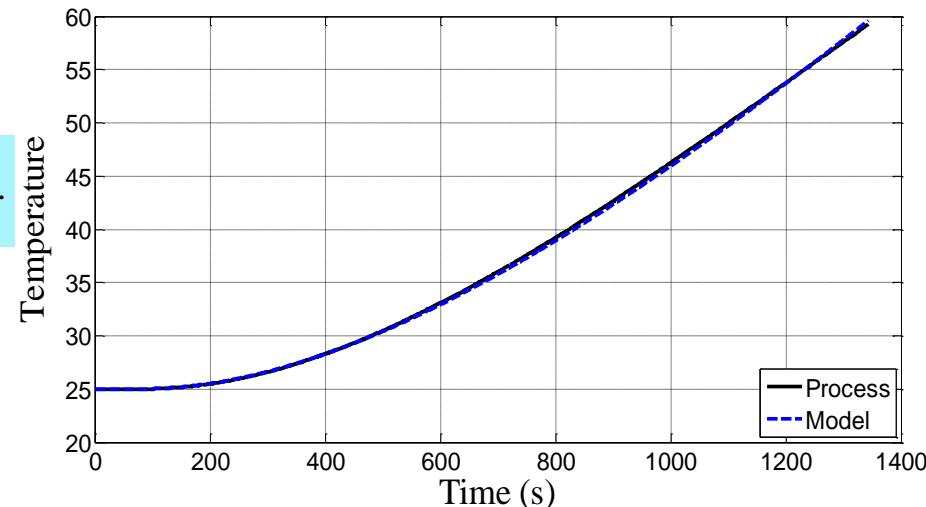
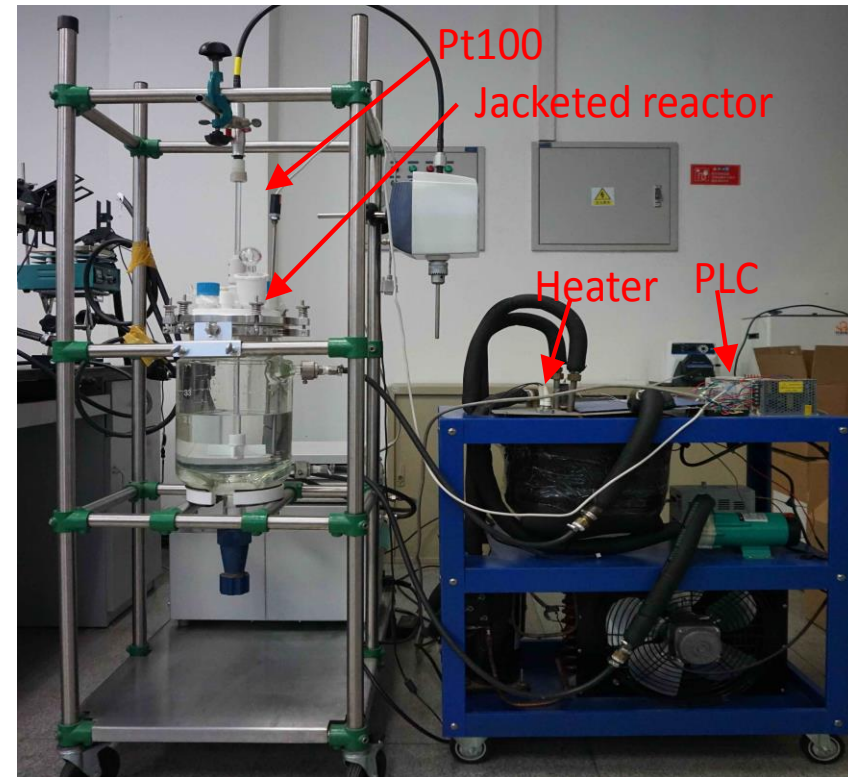
$$K(z) = \frac{(1 - \lambda_f)^4 (\beta_0 + \beta_1 z + \beta_2 z^2)}{4.9557 \times 10^{-6} z(z - 1)(z - 0.9899)}$$

$$\beta_0 = 1 - \beta_1 - \beta_2 \quad \beta_1 = 4 / (1 - \lambda_f) - \beta_2$$

$$\beta_2 = \frac{(0.9961 - \lambda_f)^4}{(0.9961 - 1)^2 (1 - \lambda_f)^4} - \frac{4}{(1 - 0.9961)(1 - \lambda_f)} - \frac{1}{(0.9961 - 1)^2}$$

Set-point filter:

$$K_f(z) = \frac{z(z - \lambda_f)^4 (1 - \lambda_s)^3}{(1 - \lambda_f)^4 (\beta_0 + \beta_1 z + \beta_2 z^2) (z - \lambda_s)^3}$$



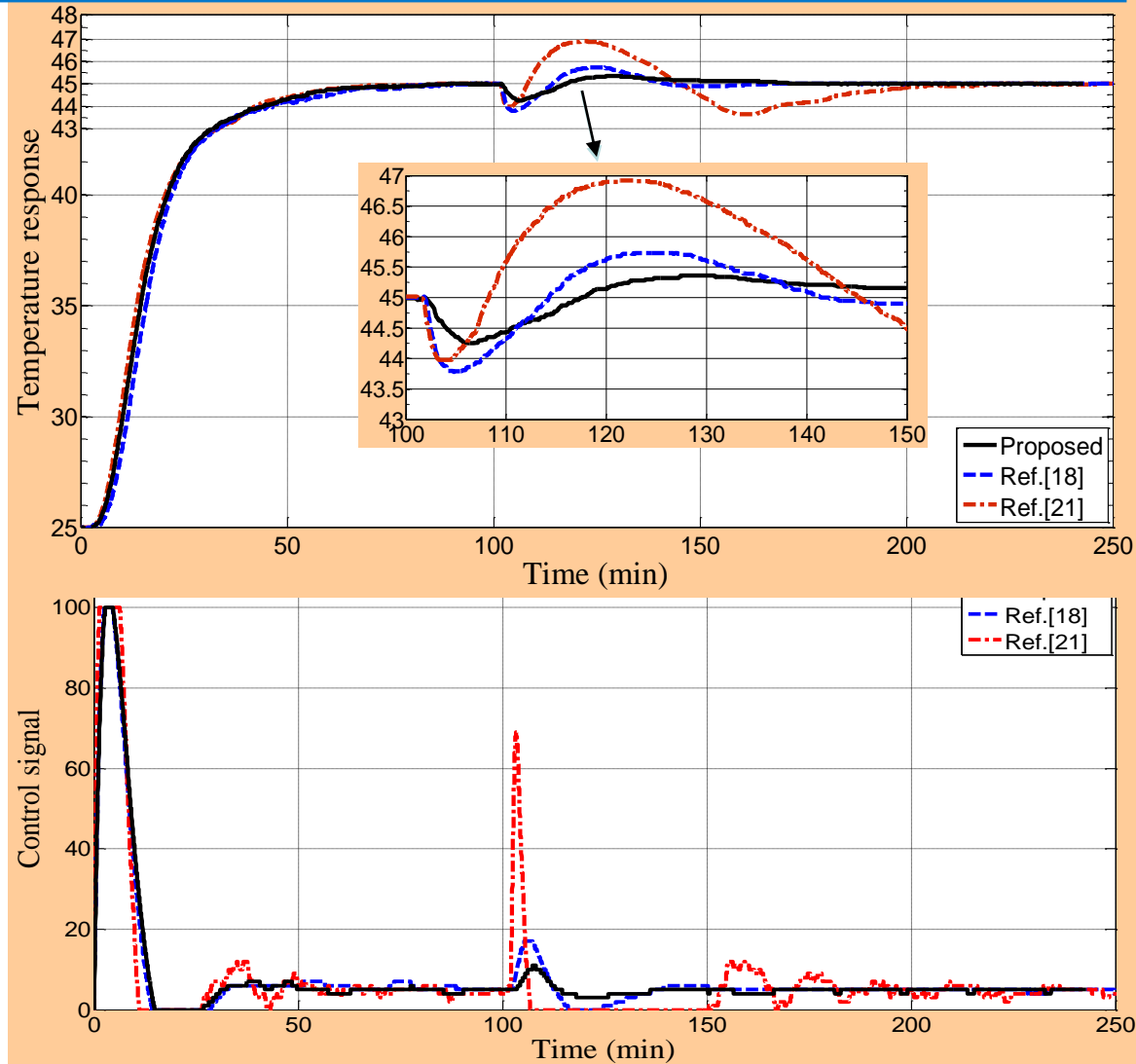
Discrete-time PID design for long time delay processes

Heat up the aqueous solution from the room temperature (25°C) to 45°C.

Load disturbance arises from feeding 200(ml) solvent of distilled water.

Input constraint: $0 \leq u \leq 100$
regulate the heating power

Compared with the filtered PID method [21] based on delayed output feedback, **more than 30 minutes** are saved for recovering the solution temperature to the operation zone of $(45 \pm 0.1)^\circ\text{C}$ against the load disturbance.



[18] Normey-Rico J.E, Camacho E.F. Unified approach for robust dead-time compensator design. *J. Process Control*, 19, 38-47, 2009.

[21] Jin, Q.B., Liu, Q. Analytical IMC-PID design in terms of performance/robustness tradeoff for integrating processes: From 2-Dof to 1-Dof. *J. Process Control*, 24(3), 22-32, 2014.

Our recent publications on discrete-time domain PID design

- ✓ Tao Liu*, Pedro García, Yueling Chen, Xuhui Ren, Pedro Albertos, Ricardo Sanz. New Predictor and 2DOF Control Scheme for Industrial Processes with Long Time Delay. *IEEE Transactions on Industrial Electronics*, 2018, 65 (5), 4247-4256.
- ✓ Yueling Chen, Tao Liu*, Pedro García, Pedro Albertos. Analytical design of a generalized predictor-based control scheme for low-order integrating and unstable systems with long time delay. *IET Control Theory & Application*, 2016, 10(8), 884-893.
- ✓ Dong Wang, Tao Liu*, Ximing Sun, Chongquan Zhong. Discrete-time domain two-degree-of-freedom control for integrating and unstable processes with time delay. *ISA Transactions*, 2016, 63, 121-132.
- ✓ Tao Liu*, Furong Gao. Enhanced IMC design of load disturbance rejection for integrating and unstable processes with slow dynamics. *ISA Transactions*, 2011, 50 (2), 239-248.
- ✓ Tao Liu, Furong Gao*. New insight into internal model control filter design for load disturbance rejection. *IET Control Theory & Application*, 2010, 4 (3), 448-460.
- ✓ Jiyao Cui, Yueling Chen, Tao Liu*. Discrete-time domain IMC-based PID control design for industrial processes with time delay. *The 35th Chinese Control Conference (CCC), Chengdu, China, July 27-29, 2016*, 5946-5951.
- ✓ Hongyu Tian, Xudong Sun, Dong Wang, Tao Liu*. Predictor based two-degree-of-freedom control design for industrial stable processes with long input delay. *The 35th Chinese Control Conference (CCC), Chengdu, China, July 27-29, 2016*, 4348-4353.

PID design for batch process optimization



Industrial batch processes, e.g., injection molding machines, chemical reactors

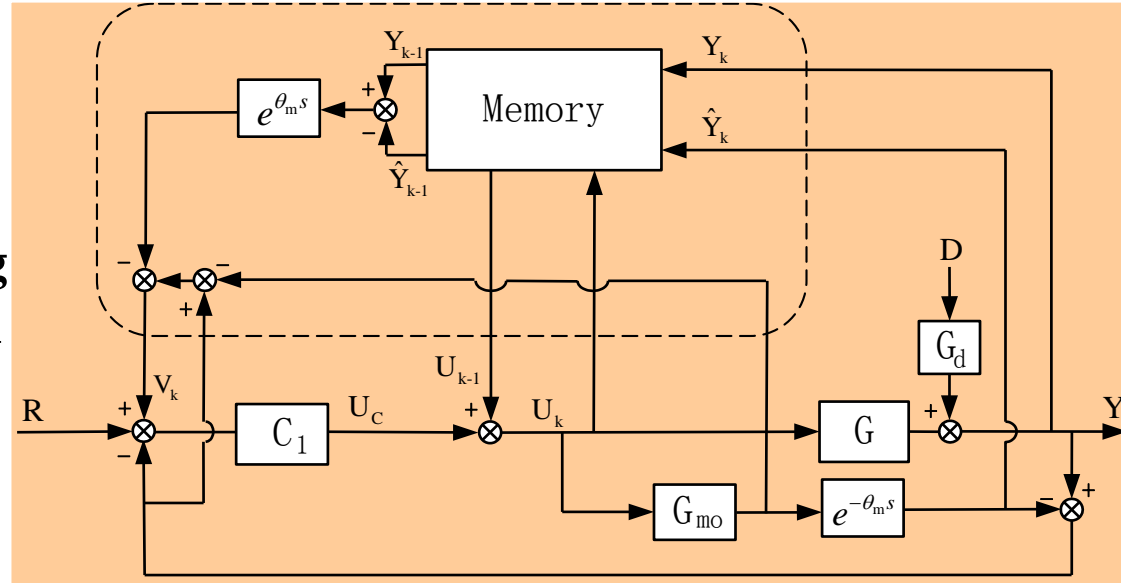


Features:

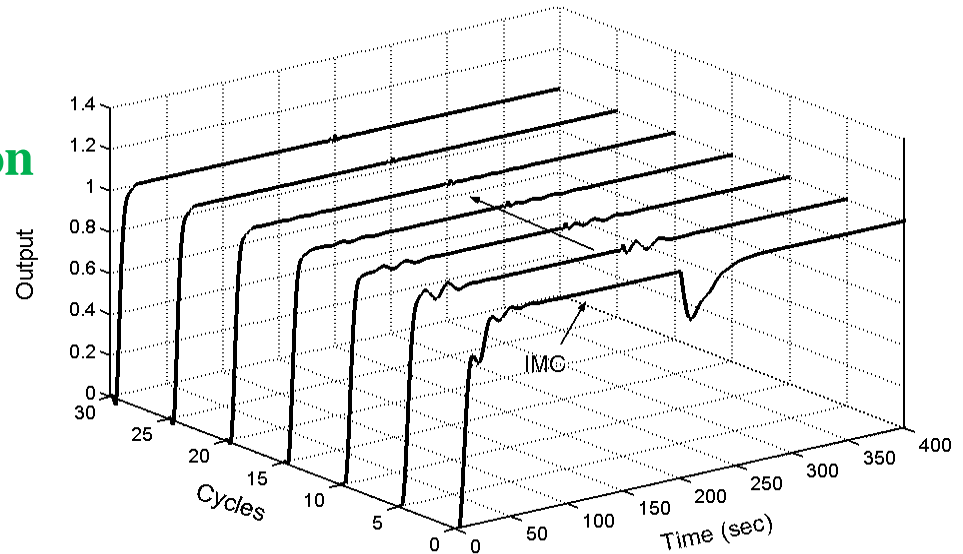
1. Repetitive operation for production;
2. Historical cycle information for progressively improving system performance;
3. Time or batch varying uncertainties.

PID design for batch process optimization

IMC-based iterative learning control (ILC) for perfect tracking using historical cycle information



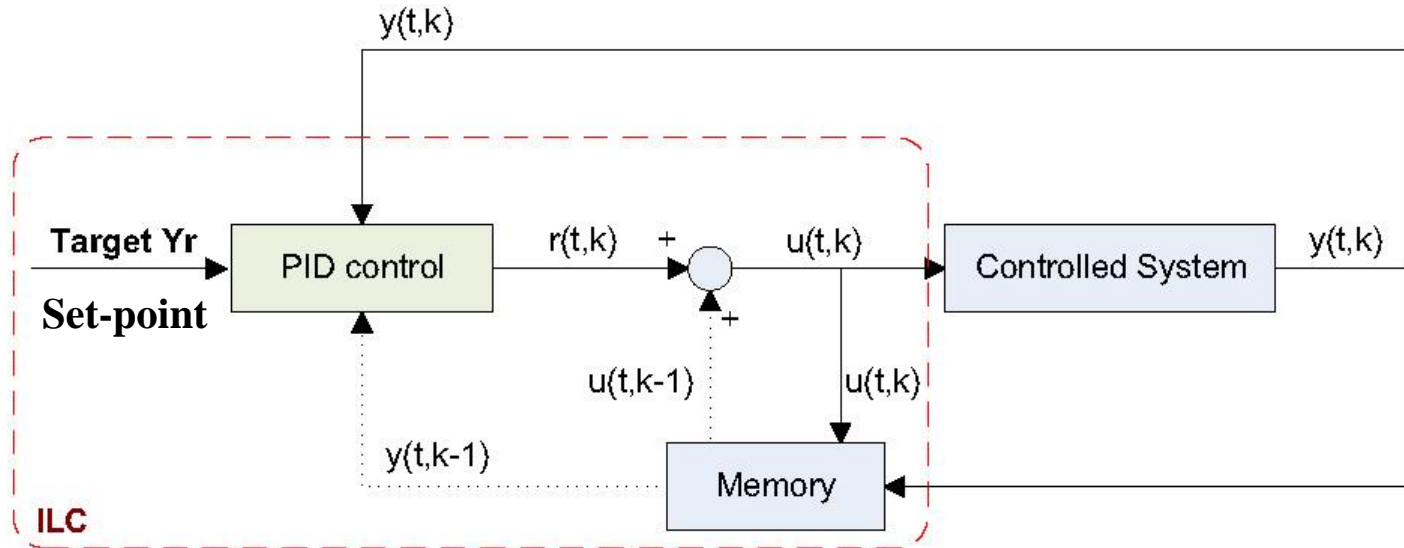
Advantage: batch control optimization



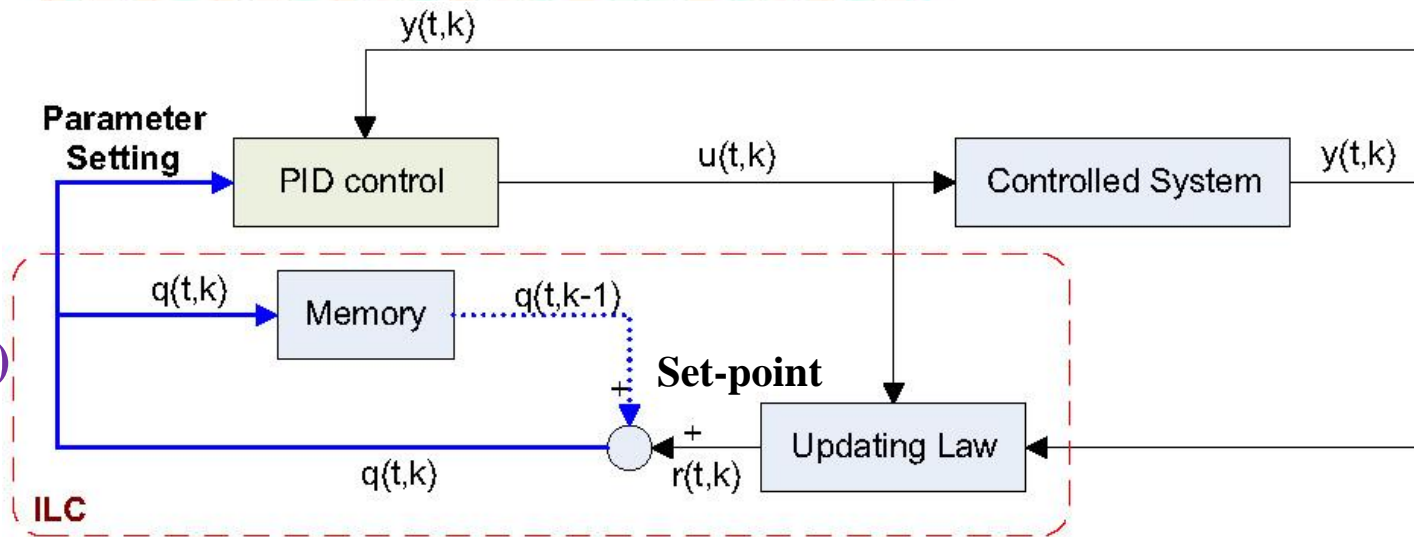
Tao Liu, Furong Gao, Youqing Wang. IMC-based iterative learning control for batch processes with uncertain time delay. *Journal of Process Control*, 20 (2), 173-180, 2010.

PID design for batch process optimization

Type I: PID-based ILC system
(modify the input error for batch operation)

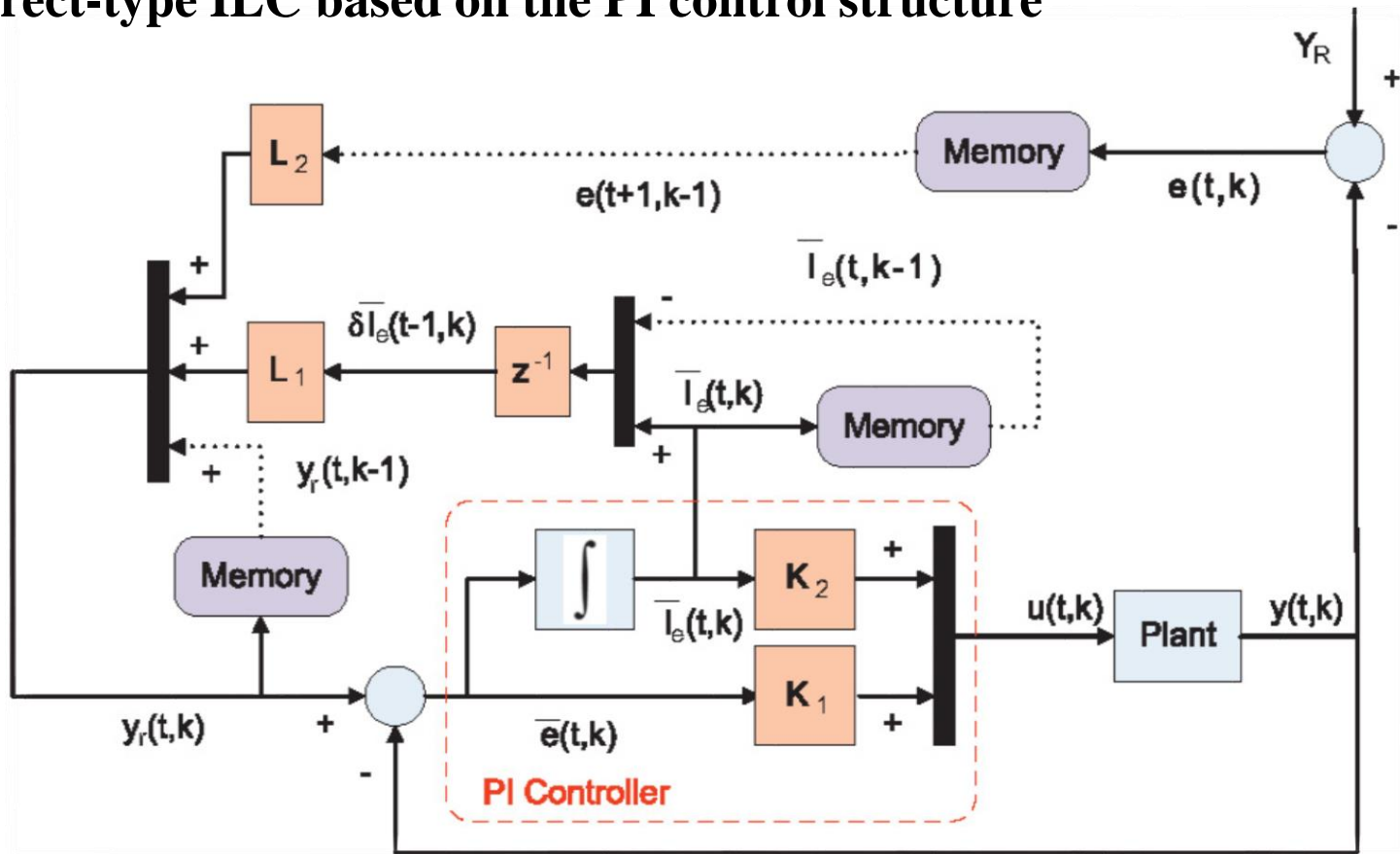


Type II: ILC-based PID control system
(controller retuning for batch operation)



PID design for batch process optimization

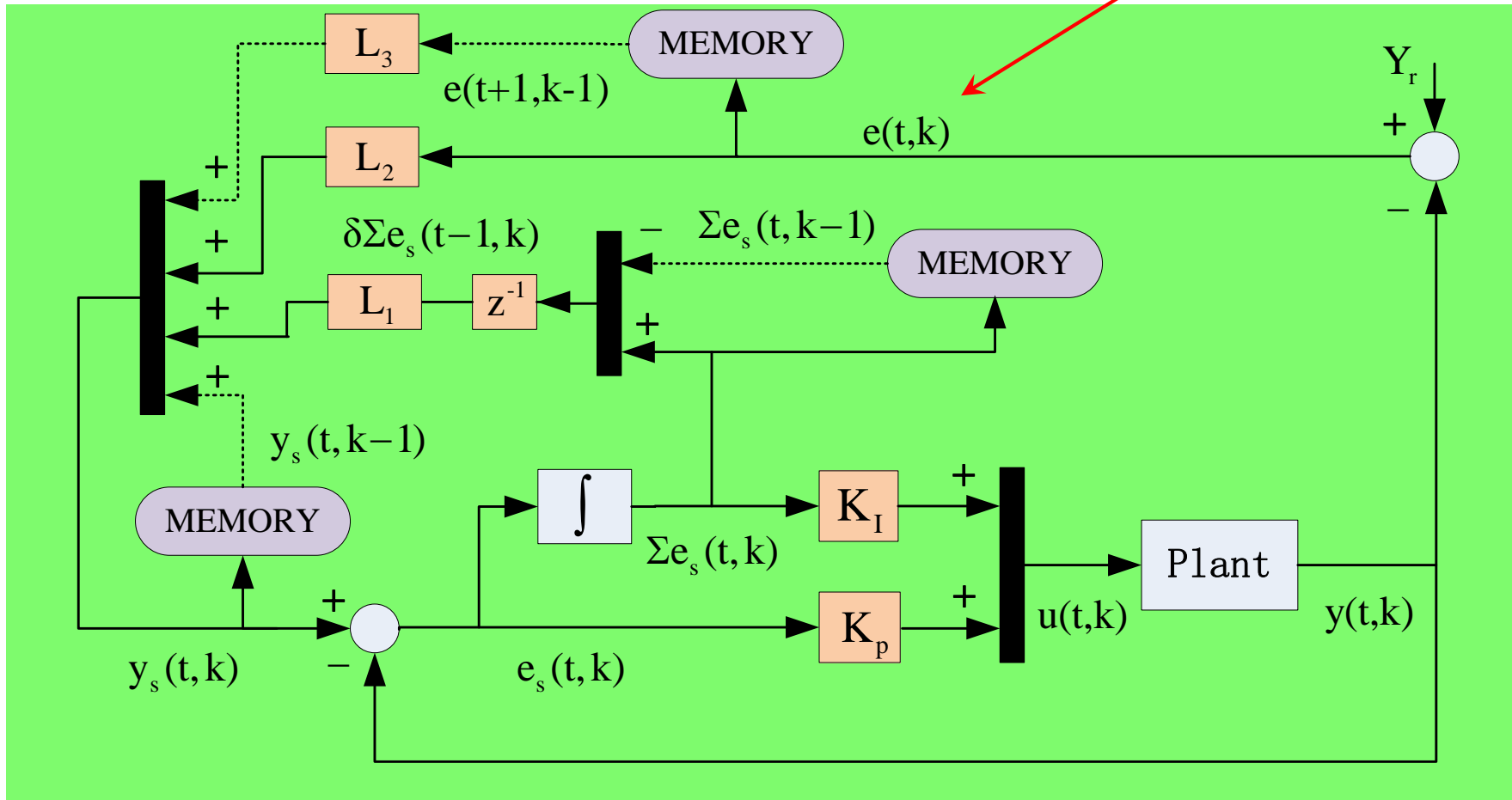
Indirect-type ILC based on the PI control structure



Advantage: No need to modify the closed-loop PI controller for ILC design, i.e., The PI controller and ILC updating law can be separately designed.

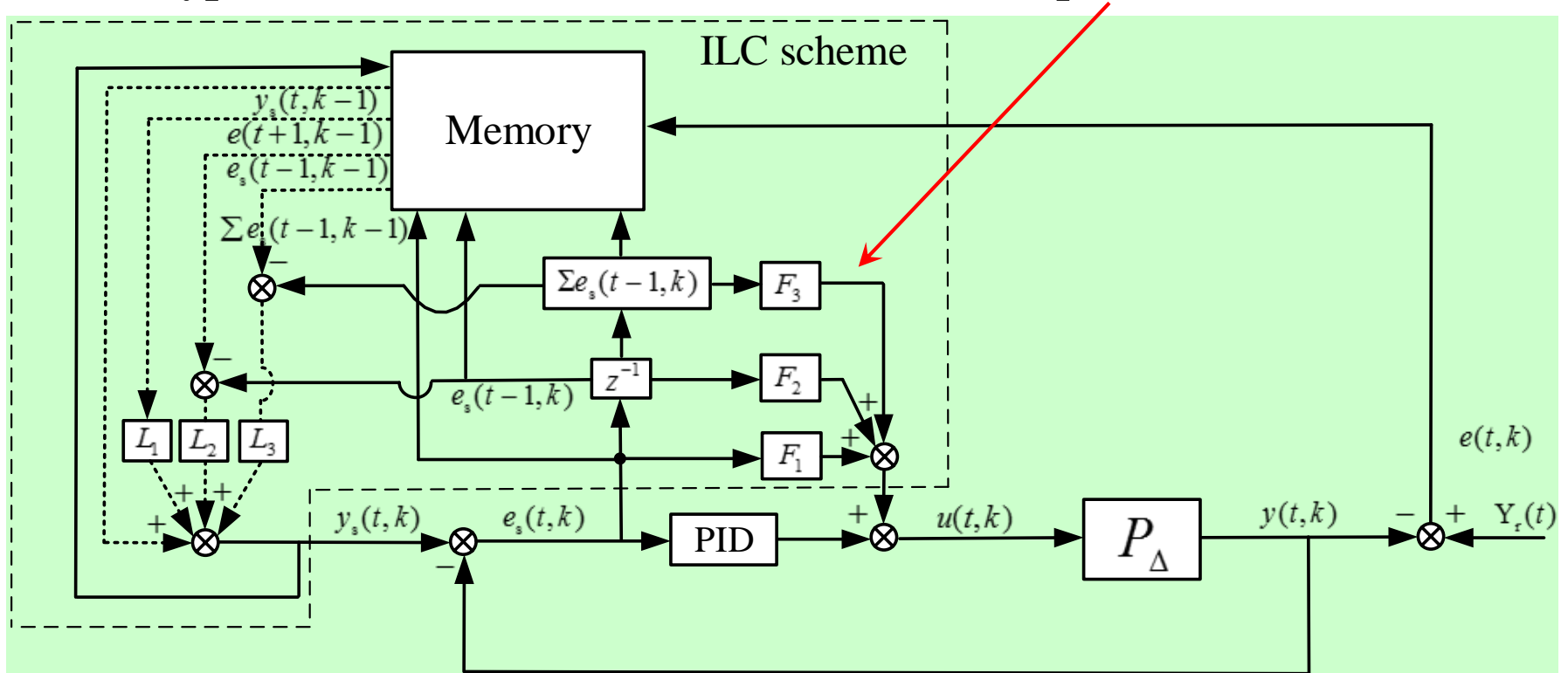
PID design for batch process optimization

Indirect-type ILC design based on the PI control structure and current output error



PID design for batch process optimization

Indirect-type ILC based on the PID control structure plus feedforward control



$$P_\Delta : \begin{cases} x(t+1, k) = [A_m + \Delta \tilde{A}(t, k)]x(t, k) + [B_m + \Delta \tilde{B}(t, k)]u(t, k) + \omega(t, k) \\ y(t, k) = Cx(t, k), \quad 0 \leq t \leq T_p; \\ x(0, k) = x(0), \quad k=1, 2, \dots. \end{cases}$$

t : time index

k : batch index

T_p : batch period

Robust PID tuning for indirect-type ILC

A PID control law in discrete-time domain

$$u_{\text{PID}}(t) = k_{\text{P}}e(t) + k_{\text{I}} \sum e(t) + k_{\text{D}}[e(t+1) - e(t)]$$

$$e(t, k) = Y_{\text{r}}(t) - y(t, k)$$

approximate $\begin{matrix} \text{L} \\ \text{red arrow} \end{matrix}$ $e(t) - e(t-1)$
 $[e(t) + e(t-2) - 2e(t-1)] / 2$

State-space closed-loop PID system description

$$\begin{cases} \begin{bmatrix} x(t+1) \\ \sum e(t) \end{bmatrix} = \begin{bmatrix} \tilde{A} + \tilde{B}[\hat{k}_{\text{D}}(CI - A_{\text{m}}) - \hat{k}_{\text{P}}C] & \tilde{B}\hat{k}_{\text{I}} \\ -C & I \end{bmatrix} \begin{bmatrix} x(t) \\ \sum e(t-1) \end{bmatrix} + \begin{bmatrix} I \\ \mathbf{0} \end{bmatrix} \omega(t) \\ y(t) = [C \quad \mathbf{0}] \begin{bmatrix} x(t) \\ \sum e(t-1) \end{bmatrix} \end{cases}$$

where

$$\tilde{A} = A_{\text{m}} + \Delta\tilde{A}(t)$$

$$\tilde{B} = B_{\text{m}} + \Delta\tilde{B}(t)$$

$$\hat{k}_{\text{P}} = (I + k_{\text{D}}CB_{\text{m}})^{-1}(k_{\text{P}} + k_{\text{I}})$$

$$\hat{k}_{\text{I}} = (I + k_{\text{D}}CB_{\text{m}})^{-1}k_{\text{I}}$$

$$\hat{k}_{\text{D}} = (I + k_{\text{D}}CB_{\text{m}})^{-1}k_{\text{D}}$$

Robust PID tuning for indirect-type ILC

The H infinity control objective for closed-loop system robust stability

$$\|e(t)\|_2 < \gamma_{\text{PID}} \|\omega(t)\|_2$$

where γ_{PID} denotes the robust performance level.

Theorem 1: The PID control system is guaranteed robustly stable if there exist $P_{11} > 0$ $P_{22} > 0$ matrices P_{12} , R_1 , R_2 , and positive scalars ε_1 , ε_2 , such that

$$\begin{bmatrix} -P + \varepsilon_1 \Phi_{A1} \Phi_{A1}^T + \varepsilon_2 \Phi_{B1} \Phi_{B1}^T & \Gamma & D_g & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ * & -P & \mathbf{0} & PH^T C^T & P\Phi_{A2}^T & P\Phi_{B2}^T \\ * & * & -\gamma_{\text{PID}} \mathbf{I} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ * & * & * & -\gamma_{\text{PID}} \mathbf{I} & \mathbf{0} & \mathbf{0} \\ * & * & * & * & -\varepsilon_1 \mathbf{I} & \mathbf{0} \\ * & * & * & * & * & -\varepsilon_2 \mathbf{I} \end{bmatrix} < 0$$

where $D_g = [\mathbf{I} \ \mathbf{0}]^T$ $H = [\mathbf{I} \ \mathbf{0}]$ $\Phi_{A1} = [\Delta \bar{A}_1^T, \mathbf{0}]^T$ $\Phi_{A2} = [\Delta \bar{A}_2 P_{11}, \Delta \bar{A}_2 P_{12}]$

$$\begin{aligned} \Phi_{B1} &= [\Delta \bar{B}_1^T, \mathbf{0}]^T \\ \Phi_{B2} &= [\Delta \bar{B}_2 R_1, \Delta \bar{B}_2 R_2] \end{aligned} \quad P = \begin{bmatrix} P_{11} & P_{12} \\ * & P_{22} \end{bmatrix} \quad \Gamma = \begin{bmatrix} A_m P_{11} + B_m R_1 & A_m P_{12} + B_m R_2 \\ -C P_{11} + P_{12}^T & -C P_{12} + P_{22} \end{bmatrix}$$

Robust PID tuning for indirect-type ILC

Correspondingly, the PID controller is determined by

$$[\hat{k}_D(CI - A_m) - \hat{k}_P C \quad \hat{k}_I] = [R_1 \quad R_2]P^{-1}$$

$$k_I = \hat{k}_I(I + k_D CB_m)$$

$$k_P = \hat{k}_P(I + k_D CB_m) - k_I$$

where k_D is user specified for implementation. If $k_D = 0$, it is a PI controller.

An optimal program for tuning PID to accommodate for the uncertainty bounds,

$$\text{Min}_{\Delta\tilde{A}(t), \Delta\tilde{B}(t)} \gamma_{\text{PID}}$$

Guideline: A smaller value of γ_{PID} leads to faster output response with a more aggressive control action, and vice versa.

Robust PI tuning for indirect-type ILC

Another robust tuning of PI controller by assigning the closed-loop system poles to a prescribed circular region, $D(\alpha, r)$ centered at $(\alpha, 0)$ with radius r and $|\alpha| + r < 1$, i.e.,

$$\lambda(\tilde{A}) \subset D(\alpha, r)$$

while the closed-loop transfer function $H(z) = \hat{C}(zI - \tilde{A})^{-1}\hat{D}$ satisfies

$$\|H(z)\|_{\infty} < \gamma_{\text{PI}}$$

Theorem 2: The PI control system is guaranteed robustly D-stable if there exist matrices $P_1 > 0$, $P_3 > 0$, $P_2 = P_2^T$, R_1, R_2 , and positive scalar ε , such that

$$\begin{bmatrix} \Lambda_1 & 0 & \Lambda_2 & P\hat{C}^T & 0 & P\hat{F}^T \\ * & -\beta_1^{-1}\gamma_{\text{PI}}^2 I & \hat{D}^T & 0 & \hat{D}^T & 0 \\ * & * & \Lambda_3 & 0 & 0 & 0 \\ * & * & * & -\beta_1^{-1}I & 0 & 0 \\ * & * & * & * & -\beta_2 P & 0 \\ * & * & * & * & * & -\varepsilon I \end{bmatrix} < 0$$

where $\beta_1 = 1 - |\alpha|$, $\beta_2 = (\beta_1^{-1} - 1)^{-1}$

$$\Lambda_1 = -\alpha P\hat{A}^T - \alpha\hat{A}P + (\alpha^2 - r^2)P + \varepsilon\alpha^2\hat{E}\hat{E}^T$$

$$P = \begin{bmatrix} P_1 & P_2 \\ P_2 & P_3 \end{bmatrix} \quad P\hat{F}^T = \begin{bmatrix} P_1 F_A^T - R_1^T F_B^T \\ P_2^T F_A^T - R_2^T F_B^T \end{bmatrix} \quad \hat{A}P = \begin{bmatrix} AP_1 - BR_1 & AP_2 - BR_2 \\ -CP_1 + P_2^T & -CP_2 + P_3 \end{bmatrix} \quad \Lambda_2 = P\hat{A}^T - \varepsilon\alpha\hat{E}\hat{E}^T$$

$$\Lambda_3 = -P + \varepsilon\hat{E}\hat{E}^T$$

Robust PI tuning for indirect-type ILC

Correspondingly, the PI controller is determined by

$$\left[(K_P + K_I)C \quad -K_I \right] = \left[R_1 \quad R_2 \right] P^{-1}$$



$$(R_1 \quad R_2)P^{-1} = (\hat{K}_P \quad \hat{K}_I)$$

$$K_I = -\hat{K}_I$$

$$K_P = \hat{K}_P C^T (C C^T)^{-1} + \hat{K}_I$$

To optimize the robust H infinity control performance, the PI controller can be determined by solving the following optimization program,

$$\underset{\Delta \tilde{A}(t), \Delta \tilde{B}(t)}{\text{Min}} \gamma_{\text{PI}}$$

Guideline: A smaller value of γ_{PI} leads to faster output response with a more aggressive control action, and vice versa.

PI based set-point learning design for indirect-type ILC

Two-dimensional (2D) system description of the indirect ILC scheme

$$\begin{cases} \begin{bmatrix} \delta x(t+1, k) \\ \delta e_s(t, k) \\ \delta \sum e_s(t, k) \\ e(t+1, k) \end{bmatrix} = \tilde{\Psi} \begin{bmatrix} \delta x(t, k) \\ \delta e_s(t-1, k) \\ \delta \sum e_s(t-1, k) \\ e(t+1, k-1) \end{bmatrix} + D_w \varpi(t) \\ \zeta(t, k) = G \begin{bmatrix} \delta x(t, k) \\ \delta e_s(t-1, k) \\ \delta \sum e_s(t-1, k) \\ e(t+1, k-1) \end{bmatrix} \end{cases}$$

where

$$\zeta(t, k) = e(t+1, k-1)$$

$$G = [\mathbf{0} \quad \mathbf{0} \quad \mathbf{0} \quad \mathbf{I}]$$

$$D_w = [\mathbf{I} \quad \mathbf{0} \quad \mathbf{0} \quad -\mathbf{C}^T]^T$$

$$\tilde{\Psi} = \begin{bmatrix} \tilde{A} - \tilde{B}(k_p + k_i + k_d + F_1)C & \tilde{B}[(k_p + k_i + k_d + F_1)L_2 + F_2 - k_d] \\ -C & L_2 \\ -C & L_2 \\ -\tilde{C}\tilde{A} + \tilde{C}\tilde{B}(k_p + k_i + k_d + F_1)C & -\tilde{C}\tilde{B}[(k_p + k_i + k_d + F_1)L_2 + F_2 - k_d] \\ \tilde{B}[(k_p + k_i + k_d + F_1)L_3 + F_3 + k_i] & \tilde{B}(k_p + k_i + k_d + F_1)L_1 \\ L_3 & L_1 \\ \mathbf{I} + L_3 & L_1 \\ -\tilde{C}\tilde{B}[(k_p + k_i + k_d + F_1)L_3 + F_3 + k_i] & \mathbf{I} - \tilde{C}\tilde{B}(k_p + k_i + k_d + F_1)L_1 \end{bmatrix}$$

PI based set-point learning design for indirect-type ILC

The control objectives for robust tracking from batch to batch

$$J_{\text{BP}} = \sum_{t=0}^{N_1=T_p} \sum_{k=0}^{N_2 \rightarrow \infty} (\gamma_{\text{ILC}}^{-1} \|\zeta(t, k+1)\|_2^2 - \gamma_{\text{ILC}} \|\varpi(t, k+1)\|_2^2) < 0$$

2D Roesser's system stability [1]:

$$\begin{cases} \begin{bmatrix} x^h(i+1, j) \\ x^v(i, j+1) \end{bmatrix} = \begin{bmatrix} A_{11} + \Delta A_{11} & A_{12} + \Delta A_{12} \\ A_{21} + \Delta A_{21} & A_{22} + \Delta A_{22} \end{bmatrix} \begin{bmatrix} x^h(i, j) \\ x^v(i, j) \end{bmatrix} + \omega(i, j) \\ y(i, j) = [C_1 \quad C_2] \begin{bmatrix} x^h(i, j) \\ x^v(i, j) \end{bmatrix} \\ i, j=0,1,2,\dots \end{cases}$$

Robust stability condition [1]:

$$\tilde{A}^T P \tilde{A} - P < 0$$

$$\tilde{A} = \begin{bmatrix} A_{11} + \Delta A_{11} & A_{12} + \Delta A_{12} \\ A_{21} + \Delta A_{21} & A_{22} + \Delta A_{22} \end{bmatrix}$$
$$P = \text{diag}\{P_1, P_2\}$$

PI based set-point learning design for indirect-type ILC

Define

$$x^h(t, k) = \begin{bmatrix} \delta x(t, k) \\ \delta e_s(t-1, k) \\ \delta \sum e_s(t-1, k) \end{bmatrix}$$

$$x^v(t, k) = e(t+1, k)$$

Lyapunov-Krasovskii function used for analyzing 2D asymptotic stability

$$\Delta V = V_{\varrho} \begin{bmatrix} x^h(t+1, k) \\ x^v(t, k) \end{bmatrix} - V_{\varrho} \begin{bmatrix} x^h(t, k) \\ x^v(t, k-1) \end{bmatrix}$$

The objective function of robust batch operation for minimization

$$J_{\text{BP}} = \sum_{t=0}^{N_1=T_p} \sum_{k=0}^{N_2 \rightarrow \infty} (\gamma_{\text{ILC}}^{-1} \|\zeta(t, k+1)\|_2^2 - \gamma_{\text{ILC}} \|\varpi(t, k+1)\|_2^2 + \Delta V) - \sum_{t=0}^{N_1=T_p} \sum_{k=0}^{N_2 \rightarrow \infty} \Delta V < 0$$

$$\begin{cases} \delta x(0, 0) = \delta x(0, 1) = \delta x(1, 0) = 0; \\ \delta e_s(0, 0) = \delta e_s(0, 1) = \delta e_s(1, 0) = 0; \\ \delta \sum e_s(0, 0) = \delta \sum e_s(0, 1) = \delta \sum e_s(1, 0) = 0; \\ e(0, 0) = e(0, 1) = e(1, 0) = 0. \end{cases}$$



$$\sum_{t=0}^{N_1=T_p} \sum_{k=0}^{N_2 \rightarrow \infty} \Delta V > 0$$

PI based set-point learning design for indirect-type ILC

Theorem 3: The 2D control system is guaranteed robustly stable with a H infinity control performance level, γ_{ILC} , if there exist $Q_1 > 0, Q_2 > 0, Q_3 > 0, Q_4 > 0$, matrices $\hat{F}_2, \hat{F}_3, \hat{L}_1, \hat{L}_2, \hat{L}_3$, and positive scalars $\varepsilon_1, \varepsilon_2$, such that

$$\begin{bmatrix} -Q + \varepsilon_1 \Omega_{A1} \Omega_{A1}^T + \varepsilon_2 \Omega_{B1} \Omega_{B1}^T & \Pi & D_w & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ * & -Q & \mathbf{0} & QG^T & P\Omega_{A2}^T & P\Omega_{B2}^T \\ * & * & -\gamma_{\text{ILC}}\mathbf{I} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ * & * & * & -\gamma_{\text{ILC}}\mathbf{I} & \mathbf{0} & \mathbf{0} \\ * & * & * & * & -\varepsilon_1\mathbf{I} & \mathbf{0} \\ * & * & * & * & * & -\varepsilon_2\mathbf{I} \end{bmatrix} < 0$$

where $Q = \text{diag}\{Q_1, Q_2, Q_3, Q_4\}$ $D_g = [\mathbf{I} \ \mathbf{0}]^T$ $H = [\mathbf{I} \ \mathbf{0}]$

$$\Omega_{A1} = [\Delta\bar{A}_1^T, \mathbf{0}, \mathbf{0}, -\Delta\bar{A}_1^T C^T]^T \quad \Omega_{A2} = [\Delta\bar{A}_2, \mathbf{0}, \mathbf{0}, \mathbf{0}] \quad \Omega_{B1} = [\Delta\bar{B}_1^T, \mathbf{0}, \mathbf{0}, -\Delta\bar{B}_1^T C^T]^T$$

$$\Omega_{B2} = \begin{bmatrix} -\Delta\bar{B}_2(k_p + k_i + k_d + F_1)C, \\ \Delta\bar{B}_2[(k_p + k_i + k_d + F_1)\hat{L}_2 + \hat{F}_2 - k_d], \\ \Delta\bar{B}_2[(k_p + k_i + k_d + F_1)\hat{L}_3 + \hat{F}_3 + k_i] \\ \Delta\bar{B}_2(k_p + k_i + k_d + F_1)\hat{L}_1 \end{bmatrix} \quad \Pi = \begin{bmatrix} A_m Q_1 - B_m(k_p + k_i + k_d + F_1)CQ_1 & B_m(k_p + k_i + k_d + F_1)\hat{L}_2 + B_m\hat{F}_2 - B_mk_dQ_2 & & & & \\ -CQ_1 & \hat{L}_2 & & & & \\ -CQ_1 & \hat{L}_2 & & & & \\ -CA_mQ_1 + CB_m(k_p + k_i + k_d + F_1)CQ_1 & -CB_m(k_p + k_i + k_d + F_1)\hat{L}_2 - CB_m\hat{F}_2 + CB_mk_dQ_2 & & & & \\ B_m(k_p + k_i + k_d + F_1)\hat{L}_3 + B_m\hat{F}_3 + B_mk_iQ_3 & B_m(k_p + k_i + k_d + F_1)\hat{L}_1 & & & & \\ \hat{L}_3 & \hat{L}_1 & & & & \\ Q_3 + \hat{L}_3 & \hat{L}_1 & & & & \\ -CB_m(k_p + k_i + k_d + F_1)\hat{L}_3 - CB_m\hat{F}_3 - CB_mk_iQ_3 & Q_4 - CB_m(k_p + k_i + k_d + F_1)\hat{L}_1 & & & & \end{bmatrix}$$

PI based set-point learning design for indirect-type ILC

Correspondingly, the PI type ILC controller is determined by

$$\begin{cases} L_1 = \hat{L}_1 Q_4^{-1} \\ L_2 = \hat{L}_2 Q_2^{-1} \\ L_3 = \hat{L}_3 Q_3^{-1} \end{cases}$$

The feedforward controller is determined by

$$\begin{cases} F_2 = \hat{F}_2 Q_2^{-1} \\ F_3 = \hat{F}_3 Q_3^{-1} \end{cases}$$

To optimize the set-point tracking performance, the PI type ILC controller can be determined by solving the following optimization program,

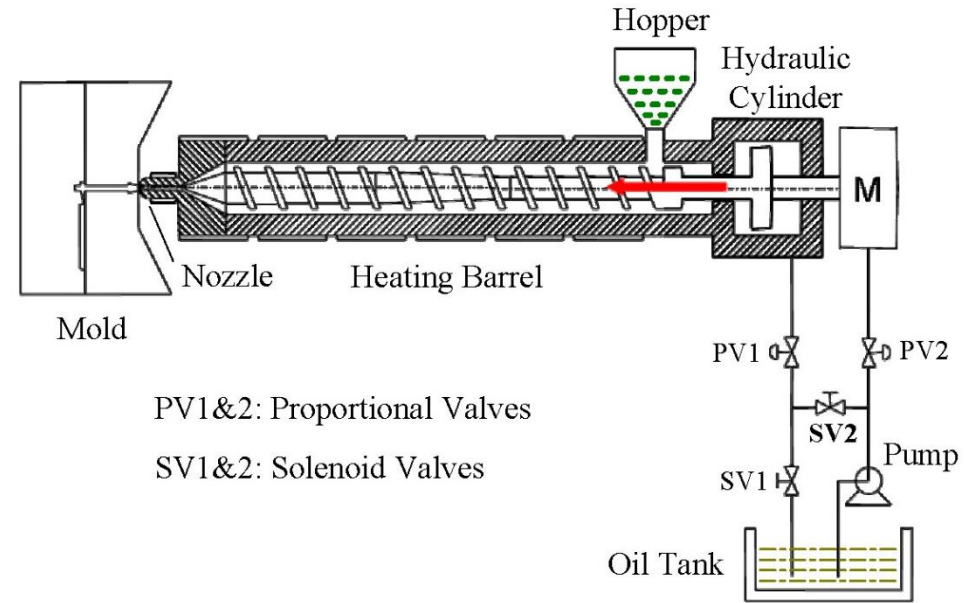
$$\text{Min}_{\Delta\tilde{A}(t), \Delta\tilde{B}(t)} \gamma_{\text{ILC}}$$

Guideline: A smaller value of γ_{ILC} leads to faster output response with a more aggressive control action, and vice versa.

PI based indirect-type ILC for batch injection molding



injection molding machine



The nozzle pressure response to the hydraulic valve input was modeled [1] by

$$y(t, k + 1) = \frac{1.239(\pm 5\%)z^{-1} - 0.9282(\pm 5\%)z^{-2}}{1 - 1.607(\pm 5\%)z^{-1} + 0.6086(\pm 5\%)z^{-2}} u(t, k + 1) + \omega(t, k + 1)$$

PI based indirect-type ILC for batch process optimization

Equivalently, the process model is rewritten in a state-space form

$$\begin{cases} x(t+1, k+1) = \begin{bmatrix} 1.607 & 1 \\ -0.6086 & 0 \end{bmatrix} x(t, k+1) + \begin{bmatrix} 1.239 \\ -0.9282 \end{bmatrix} u(t, k+1) + \begin{bmatrix} 1 \\ 0 \end{bmatrix} \omega(t, k+1) \\ y(t, k+1) = [1, 0] x(t, k+1) \end{cases}$$

**time-varying
uncertainties**

$$\Delta \tilde{A}(t) = \begin{bmatrix} 0.0804\delta(t) & 0 \\ -0.0304\delta(t) & 0 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \delta(t) & 0 \\ 0 & \delta(t) \end{bmatrix} \begin{bmatrix} 0.0804 & 0 \\ -0.0304 & 0 \end{bmatrix}$$

$$\Delta \tilde{B}(t) = \begin{bmatrix} 0.062\delta(t) \\ -0.0464\delta(t) \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \delta(t) & 0 \\ 0 & \delta(t) \end{bmatrix} \begin{bmatrix} 0.062 \\ -0.0464 \end{bmatrix}$$

$$|\delta(t)| \leq 1$$

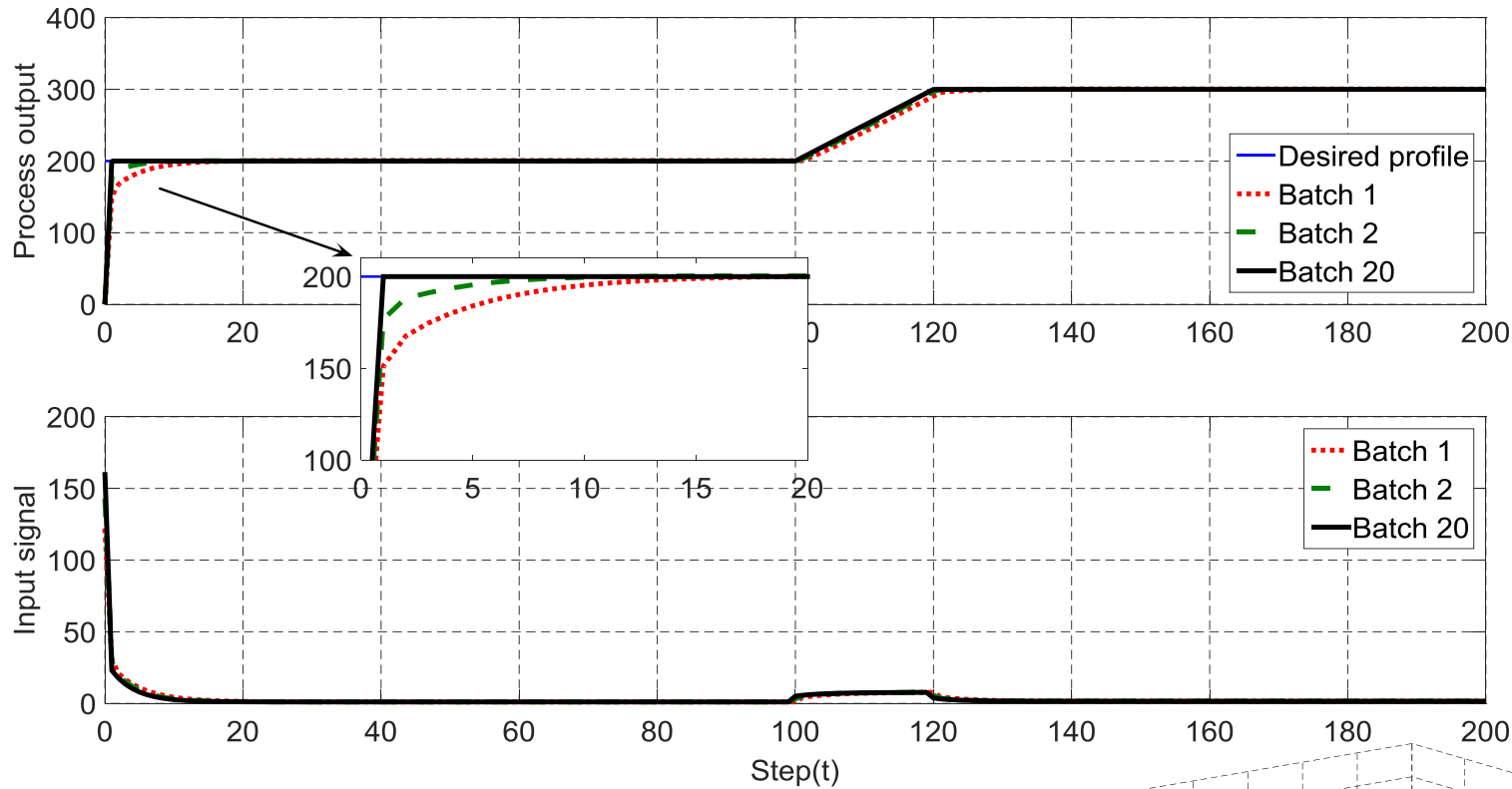
Robustly tuned PI controllers: $k_p = 1.2889$ $k_i = 0.0336$

$$F_1 \in [-1.3, 0.1] \Rightarrow \lambda[A_m - B_m(k_p + k_i + k_d + F_1)C] < 1$$

ILC controllers: $F_2 = 0$ $F_3 = -0.0097$

$$L_1 = 0.1776 \quad L_2 = 0 \quad L_3 = -0.029$$

PI based indirect-type ILC for batch process optimization

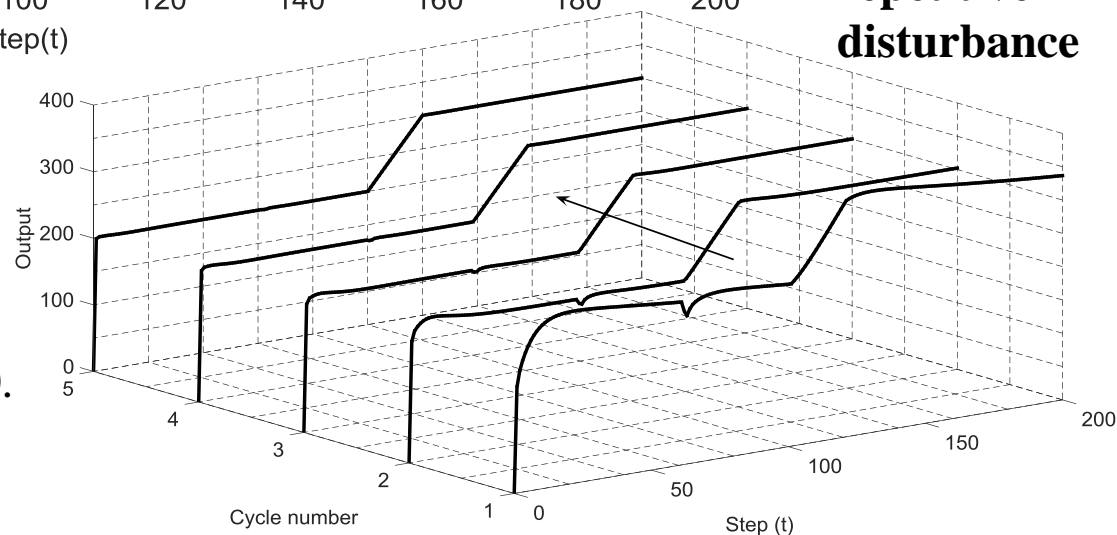


Case 1:
time-invariant
uncertainties

Case 2:
repetitive
disturbance

Desired output profile

$$Y_r = \begin{cases} 200, & 0 \leq t \leq 100; \\ 200+5(t-100), & 100 < t \leq 120; \\ 300, & 120 < t \leq T_p = 200. \end{cases}$$



PI based indirect-type ILC for batch process optimization

Case 3: Time-varying uncertainties

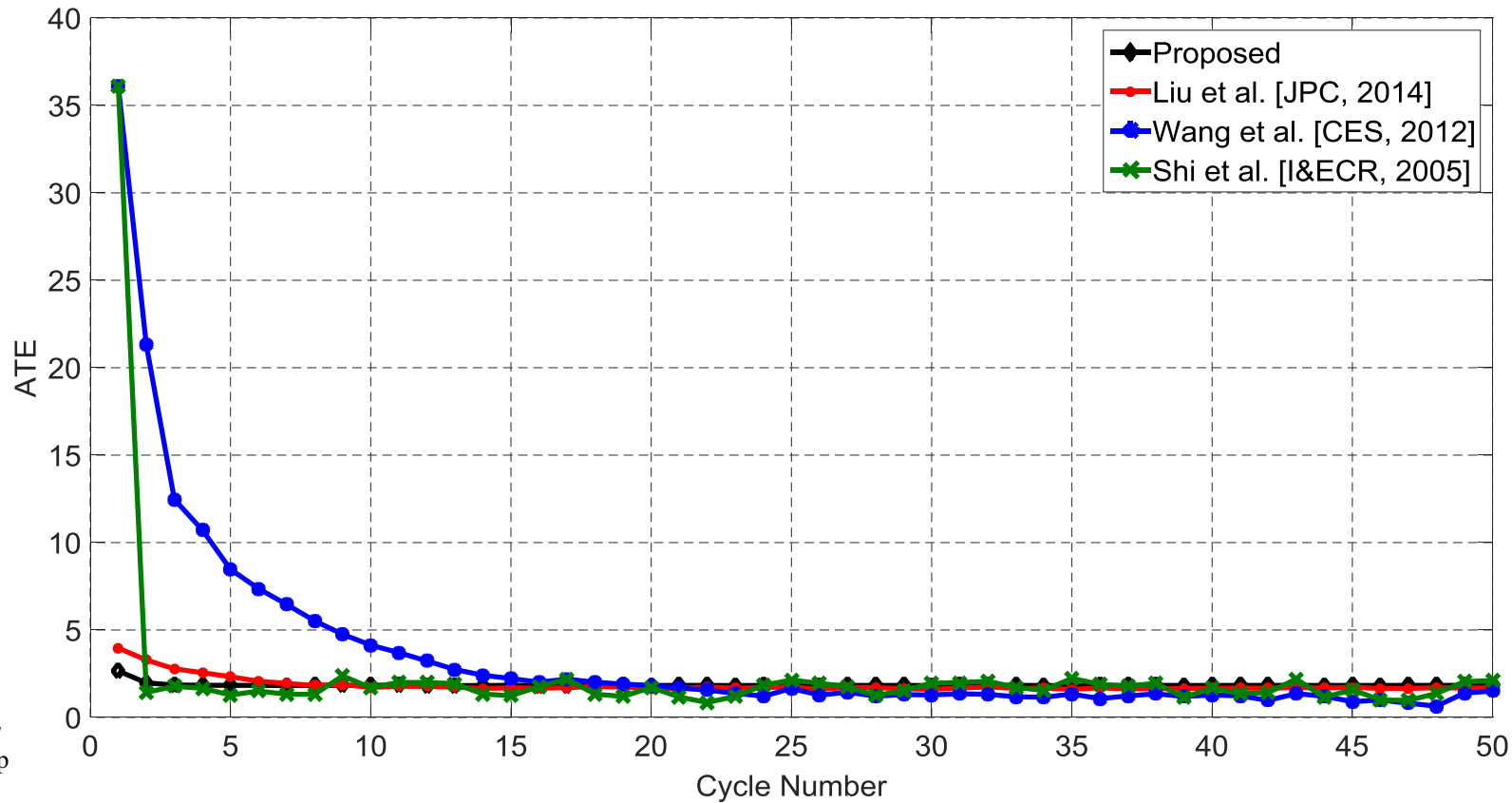
$$|\delta(t)| \leq 0.1$$

$$\omega(t, k+1) = \sin(t + \theta(k))$$

$$\theta(k) \in [0, 2\pi]$$

$$\text{ATE}(k)$$

$$= \sum_{t=1}^{T_p} |e(t, k)| / T_p$$



Plot of the output error for batch operation

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- [1] **Tao Liu**, Xue Z. Wang, Junhui Chen. Robust PID based indirect-type iterative learning control for batch processes with time-varying uncertainties. *Journal of Process Control*, 24 (12), 95-106, 2014.
 - [2] Y. Wang, **Tao Liu**, Z. Zhao. Advanced PI control with simple learning set-point design: Application on batch processes and robust stability analysis. *Chemical Engineering Science*, 71 (1), 153-165, 2012.
 - [3] Shi, J., Gao, F., Wu, T.-J. Integrated design and structure analysis of robust iterative learning control system based on a two-dimensional model. *Ind. Eng. Chem. Res.*, 44, 8095-8105, 2005.

Main Results:

- Analytical PID design in discrete-time domain for sampled control systems
- 2DOF control structure based PID design for improving disturbance rejection
- Predictor-based PID design for long time delay systems
- Robust PID tuning methods with respect to the system uncertainty bounds
- PI based indirect type ILC design for batch process optimization

Outlook:

- Data-driven PID tuning for sampled control systems
- Fractional-order PID design & PID scheduling for nonlinear systems
- PID +Memory for learning/intelligent control of industrial batch processes, repetitive systems, and robots etc.

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Pedro Albertos



Furong Gao



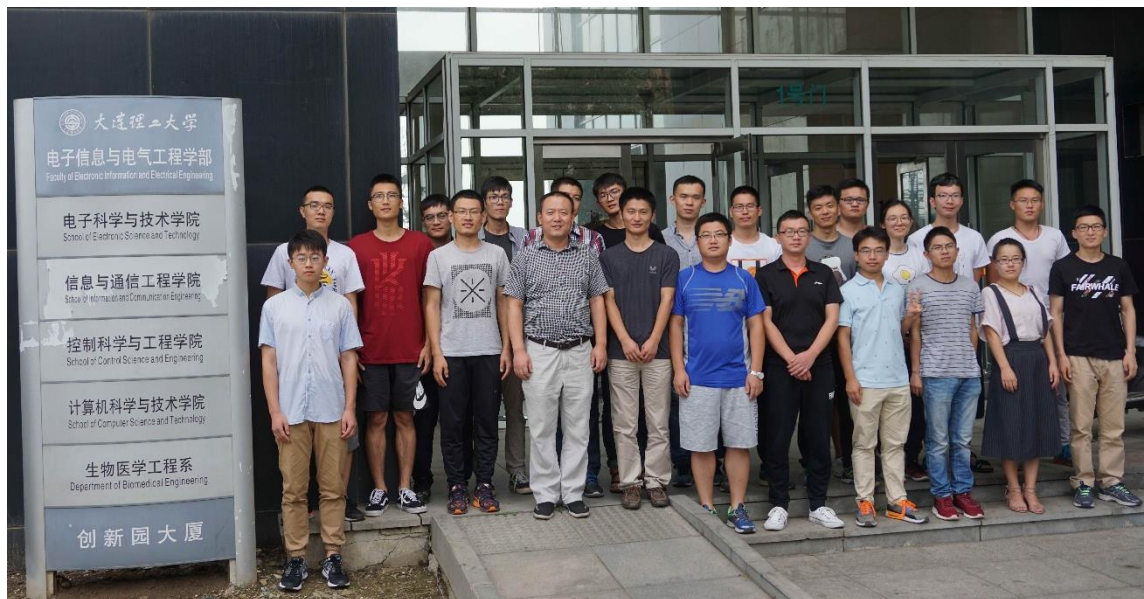
Pedro García



Wojciech Paszke



Youqing Wang



**My group photo
in 2017**



Research interests:

- Advanced control system design & system optimization;
- On-line monitoring, control design and control optimization of chemical production batch process;
- Real-time model predictive control and optimization of crystallization & drying processes;
- Design of in-situ measurement and control devices;
- PBM and CFD modeling of crystallization processes.

Thanks for your attention & comments!



Lab website: <http://act.dlut.edu.cn/>

